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FACULTY OF MECHANICAL ENGINEERING  
INSTITUTE OF AEROSPACE ENGINEERING

## **LOADS CALCULATION, STRESS ANALYSIS AND BIRD STRIKE SIMULATION OF A COMPOSITE WING LEADING EDGE**

**VÝPOČET ZATÍŽENÍ, ANALÝZA PEVNOSTI A SIMULACE NÁRAZU PTÁKA PRO  
KOMPOZITNÍ NÁBĚŽNÉ HRANY KŘÍDLA**

**DIPLOMOVÁ PRÁCE**  
MASTER'S THESIS

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## **ZADÁNÍ DIPLOMOVÉ PRÁCE**

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Ředitel ústavu Vám v souladu se zákonem č.111/1998 o vysokých školách a se Studijním a zkušebním řádem VUT v Brně určuje následující téma diplomové práce:

### **Výpočet zatížení, analýza pevnosti a simulace nárazu ptáka pro kompozitní náběžné hrany křídla**

v anglickém jazyce:

### **Loads calculation, stress analysis and bird strike simulation of a composite wing leading edge**

Stručná charakteristika problematiky úkolu:

Na základě daných konstrukčních koncepcí náběžné hrany křídla by měly být vybrány geometrické parametry a vrstvení laminátu pro konstrukční prvky s ohledem na dané zatěžovací případy a materiály. Nejzávažnější požadavek na konstrukci je přežít po nárazu ptáka případ zatížení "Get Home". Z tohoto důvodu, aby bylo možné odhadnout poškození konstrukce způsobené ptákem, měla by být provedena simulace nárazu ptáka.

Cíle diplomové práce:

S ohledem na uvedeny požadavky a materiály stanovte:

1. Geometrické parametry konstrukce slotu.
2. Vrstvení laminátu pro konstrukční prvky.

Seznam odborné literatury:

- [1] CS-25. Certification Specifications on Large Aeroplanes, 2003.
- [2] M. Chung-Yung Niu. "Aircraft structural design", 2nd edition. Hong Kong Conmilit Press Ltd. 1999. ISBN 962-7128-09-0.
- [3] N.N. Churnaieva, M.G. Iefimova, V.P. Solonin. "Wing and means for improving take-off and landing characteristics of aircraft". Moscow. Moscow state technical university of civil aviation. 2000.
- [4] M. Guida. „Study, Design and Testing of Structural Configurations for the bird - strike compliance of aeronautical components“. A thesis submitted for the Doctoral Degree in Aerospace Structure. Department of Aerospace Engineering, University of Naples "Federico II". December 2008.
- [5] S. Georgiadis, A.J. Gunnion, R.S. Thomson and B. K. Cartwright, "Bird-strike Simulation for Certification of the Boeing 787 Composite Moveable Trailing Edge", Composite Structures Journal, vol. 86, nos. 1-3, 2008, pp. 258-268. (<http://www.sciencedirect.com/science/article/pii/S0263822308000883>).
- [6] Cheng-Ho Tho, M.R. Smith. "Accurate bird strike simulation methodology for BA609 tiltrotor". Journal of the American Helicopter Society, 12/2010; 56(1):12007-1-12007-10. DOI:10.4050/JAHS.56.012007.
- [7] L.B.Lucy. „A Numerical Approach to the Testing of the Fission Hypothesis“. Astronomical Journal, Vol. 82, 1977, pp.1013-20.
- [8] "SIMULIA Abaqus User's manual". <http://www.3ds.com/support/documentation/users-guide/>.
- [9] S. Heimbs. „Bird Strike Simulations on Composite Aircraft Structures“. 2011 SIMULIA Customer Conference, Barcelona, Spain, May 17-19. 2011.

Vedoucí diplomové práce: Volodymyr Symonov, M.Sc.

Termín odevzdání diplomové práce je stanoven časovým plánem akademického roku 2014/2015.

V Brně, dne 19.11.2014

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## **Abstrakt**

Tato diplomová práce se zaměřuje na stanovení geometrických a materiálových parametrů konstrukce slotu letadla B737-200. V této práci je vypracovaný návrh kompozitového slotu s ohledem na dané zatěžovací případy, včetně tzv. "bird strike". Analytické výpočty jsou ověřené metodou konečných prvků (MKP) v programech MSC.Nastran/Patran, MSC.Dytran.

## **Klíčová slova**

konstrukce slotu, MKP, kompozitní materiál, bird strike

## **Abstract**

This thesis deals with design of slat geometrical and material parameters of the B737-200 aircraft. In this thesis there is created design of the composite slat with respect to a given load cases including bird strike. Analytical calculations are verified by FE analysis in MSC.Nastran/Patran, MSC.Dytran software.

## **Keywords**

slat construction, FEM, composite material, bird strike



## **Bibliographic citation**

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## **Declaration of authenticity**

I declare that I have elaborated my master thesis on the theme of “Loads calculation, stress analysis and bird strike simulation of a composite wing leading edge” independently, under the supervision of the master’s thesis supervisor and with the use of technical literature and other sources of information which are all quoted in the thesis and detailed in the list of literature at the end of the thesis.

In Brno 05.06.15

.....  
Raman Zubrytski



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## 1. Introduction

Composite structures are increasingly being used within the aircraft industry, even for primary structures. Same time, the application of composites within impact-endangered areas is very limited.

Collision of airplanes and birds presents a potentially hazardous situation, which is becoming much more frequent due to the ever-increasing air traffic and changes in the migration routes of bird flocks [1]. Compared to metallic structures, energy absorption and damage mechanisms of composite materials are far more complex and depend on a number of parameters such as fiber and matrix properties, ply layup, total number of layers, interfacial properties and bonding strength, impactor geometry, impact velocity and impactor initial energy. Difficulty is that high performance composites only can afford plastic strain of about 2-3% while recent aluminum alloys around 20%.

Aircraft leading edges must be certified for a proven level of bird impact resistance. In particular, the main structural requirement is to protect the torsion box and control devices from any significant damage caused by bird strike in order to allow the aircraft to land safely. In more details, certification requirements are shown in chapter 1.1.

The primary subject of the **present paper** is to develop a typical large transport airplane **flap structure** composite slat and its damage assessment according to certification requirements.

### 1.1 CS-25 Bird Strike Requirements [2]

CS 25.631 requires that the aeroplane must be designed to assure capability of continued safe flight and landing of the aeroplane after impact with a 4 lbs bird when the velocity of the aeroplane (relative to the bird along the aeroplane's flight path) is equal to  $V_c$  at sea-level or  $0.85 V_c$  at 8000 ft, whichever is the more critical. The phrase "continued safe flight and landing" in this respect may be interpreted in different ways and the effects of bird strike are addressed in various other sections of CS-25:

- (a) CS 25.571(e) which requires that the aeroplane must be capable of successfully completing a flight during which likely structural damage occurs as a result of bird impact as specified in CS 25.631. The AMC to 25.571 (in paragraph 2.7.2) specifies the loads associated with "get home" conditions that have to be met for this case;
- (f) AMC 25.1309(b) where bird strike is identified as a Particular Risk requiring investigation as part of the Common Cause Analysis.

(1) Initially all areas/zones of the aircraft prone to bird strike should be considered, either pressurized or non-pressurized, either primary or secondary structure. This would normally include areas/zones such as:

- (g) Wings (leading edges (including slats), trailing edges (flaps));

For high lift devices (flaps and slats) instead of using  $V_c$  at sea-level or  $0.85 V_c$  at 8000 ft, the appropriate maximum design speed (as per CS 25.335(e)) may be taken as the basis for determining the bird impact damage. For landing gears, the appropriate maximum speed (as per CS 25.1515) may be taken as the basis for determining the bird impact damage.

(2) Showing that under the conditions of CS 25.631 no bird penetration and no part loss occurs in the aircraft areas/zones where bird strike is of concern, is the preferred certification approach. For this scenario, continued safe flight and landing should be further substantiated considering the following effects:

- (a) Bird-strike induced deformation of structures on internal structural items, such as instrument panels or avionics racks;
- (b) Bird-strike induced deformation of structures on underlying items, systems and equipment, or on operational approved performance (corrective pilot action may be considered);and
- (c) Bird-strike induced accelerations on items, systems and equipment.

(3) If contrary to item (2) above, bird penetration and/or part loss does occur in the aircraft areas/zones where bird strike is of concern, the following should be considered:

- (a) The effects of subsequent impacts on items, systems and equipment after penetration should not prohibit continued safe flight and landing;
- (f) For bird penetration into the fuel tanks (e.g. through wing leading edge and front spar)  
it must be substantiated that fire or other hazards (e.g. the resulting fuel imbalance or the inability to continue the normal flight) would not preclude continued safe flight and landing. Fuel tank leaks due to bird strike in the vicinity or upstream path of heat sources (landing gears, engines) would normally not be considered acceptable;
- (g) The effects on continued safe flight and landing of damage and subsequent release of debris resulting from bird impact should also be addressed, for example for flaps, landing gear doors and large antennas. The effects of such parts loss should not prohibit continued safe flight and landing. This evaluation should include the effect of any debris impacting other parts of the aircraft (e.g. empennage area or engines) and should consider any hazardous asymmetric conditions arising. The use of design features such as multiple attachment points, the application of engineering judgement and the review of relevant service experience may be used to support this evaluation.

## **2 Material model**

### **2.1 Material models description**

There are numbers of finite element (FE) codes used to predict the dynamic damage of composite materials, such as LS-DYNA, ABAQUS Explicit, RADIOSS, and PAM-CRASH, which use composite material models to define the elastic, failure, and post-failure behavior of the elements. These material models account for physical properties of the material that can be measured by experiment (such as strength, modulus, and strain-to-failure) but also include software specific parameters, which either have no physical meaning or cannot be determined experimentally. Usage of non-physical parameters thus requires extensive calibration and tweaking of these material models in order to reach an agreement between experiment and simulation [3].

There are a bunch of material models such as MAT22, MAT54, and MAT55 (use a ply discount method to degrade elastic material properties), MAT58, MAT158, and MAT162 (use continuum damage mechanics to degrade the elastic properties after failure), MAT8 are available. For future comparison have been chosen MAT8, MAT54, and MAT58. Short description of each is shown below.

#### **2.1.1 MAT8 material model**

MAT8 (MSC.NASTRAN Orthotropic Elastic) material model is used for describing the elasticity characteristics of an anisotropic material, assuming linear orthotropic behavior. The accuracy of this model depends on the assumption that it will stay within the bounds of the linear portion of the stress-strain curve. This model represents a suitable approximation to modeling thin anisotropic parts such as fiber-filled injection molded products/components [4].

#### **2.1.2 MAT54 material model**

The LS-DYNA MAT54 material model is of interest for large full-scale structural damage simulations because it is a relatively simple material model with minimal input parameters. Not only does this reduce the computational requirement of a simulation, it also reduces the difficulty and amount of material testing necessary to generate the input parameters [3].

#### **2.1.3 MAT58 material model**

MAT58 material model is so called elastic damage model, where it is assumed that the deformation introduces micro cracks and cavities into the material. These defects cause primarily stiffness degradation with rather small permanent deformations unless the material undergoes rather high loading and is not close to deterioration. The main difference to MAT54 material model lies in the smooth increase of damage; no sudden change of material behavior occurs which appears more correct [5] [6].

## 2.2 Numerical simulations

Chosen material models (MAT8, MAT58) behavior were simulated in Dytran software and compared to each other and to the MAT54 material model. The main criteria for the models comparison was the minimum mistake in elastic and strength properties and elastic energy in relation to the experimental data. The assessment of the LS-DYNA MAT54 material model was done by M. Osborne in [3] using the same criteria. There was used Toray T700GC-12K-31E/#2510 [3] unidirectional (UD) carbon-epoxy tape which properties are shown below in Tab. 1. Experimental tests included uniaxial tension and compression specimens about the two material axes. Simulations were done by methodology presented in [3]. The current assessment is aimed to comparison of MAT8, MAT58 simulation to MAT54 behavior studies and experimental tests.

Matl. Dir.	Property	Units	Tension			Compression		
			AGATE	MIL-HDBK-17	DYNA	AGATE	MIL-HDBK-17	DYNA
1/ Fiber	Strength	ksi	314	319	319	210	213	213
	Modulus	Msi	18.1	18.4	18.4	16.3	16.5	18.4
	Strain	u in/in	17,366	17,337	17,400	12,846	12,909	11,600
2/ Matrix	Strength	ksi	7.09	7.09	7.09	28.8	28.8	28.8
	Modulus	Msi	1.22	1.22	1.22	1.22	1.47	1.22
	Strain	u in/in	5,813	5,811	24,000	23,618	19,592	24,000

Table 1 Toray T700GC-12K-31E/#2510 UD Tape Properties

### 2.2.1 LS-DYNA MAT58 material model

Stress-strain and energy-strain curves were taken from tests data [3] for L (longitudinal), T (transverse), and CP ( $\pm 45^\circ$  cross-ply) case.

#### 2.2.1.1 Simulation conditions

All laminates were defined using the “Shell laminate properties” menu input card, which accepted material, thickness, and orientation (angle) on a ply-by-ply basis. The Nastran CQUAD4 shell element was used for all simulations. Each laminate was subjected to tension and compression along perpendicular loading axes (i.e. longitudinal and transverse). Rather than modifying either the element connectivity or applied loading, transverse loading was accomplished by rotating a laminate’s plies by 90 degrees. In addition to the longitudinal and transverse boundary conditions shown in Figure 1, all out-of-plane displacements were constrained (Z-axis in the global Coordinate System (CS)).

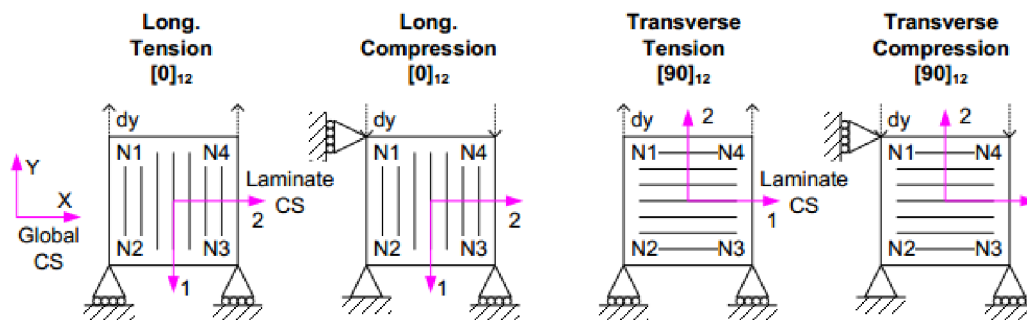


Figure 1 Single Element Mesh, Boundary Conditions, & Loading

The cross-ply simulations began with the input deck from the UD simulations of the first section. This included the MAT58 material model, loading, boundary conditions, and element geometry. The primary difference for the 12-ply cross-ply laminate was the individual ply-orientations. The loading and boundary conditions are shown on Figure 2; compressive simulations required off-axis (lateral) supports to prevent unconstrained lateral displacement.

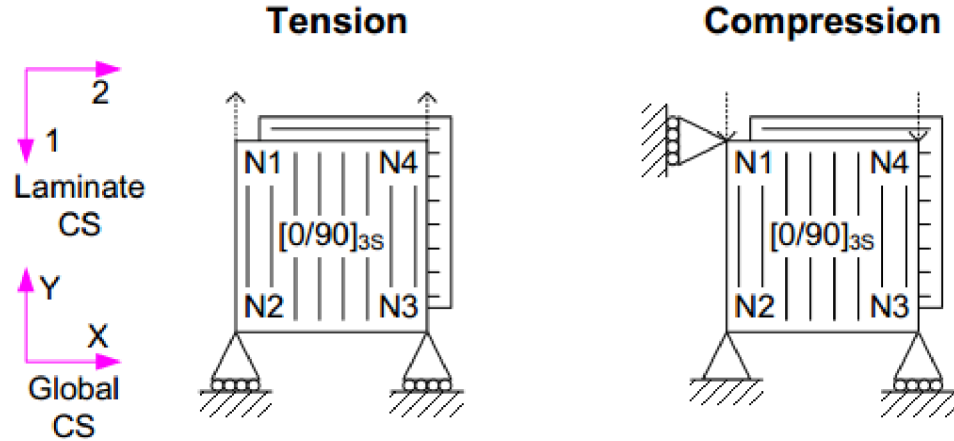


Figure 2 Cross-Ply Simulations – Loading and Boundary Conditions

### 2.2.1.2 Results

In order to get the least possible difference between experimental tests and simulations material (elastic modulus, shear modulus, ultimate failure, etc.) and special parameters (element deleting time step, max. effective strain for failure, etc.) have been selected.

Results from the  $[0]_{12}$ ,  $[90]_{12}$  and cross-ply coupon tests.

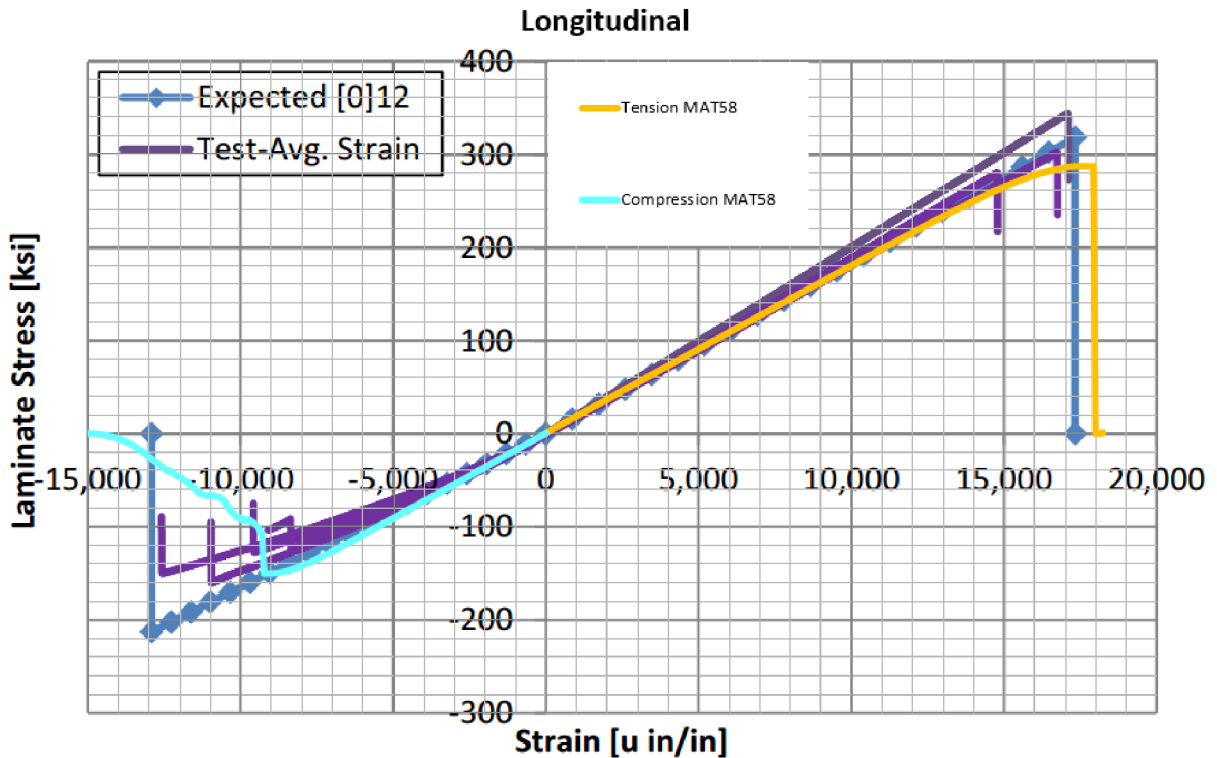


Figure 3 Stress vs. Strain – MAT58, longitudinal

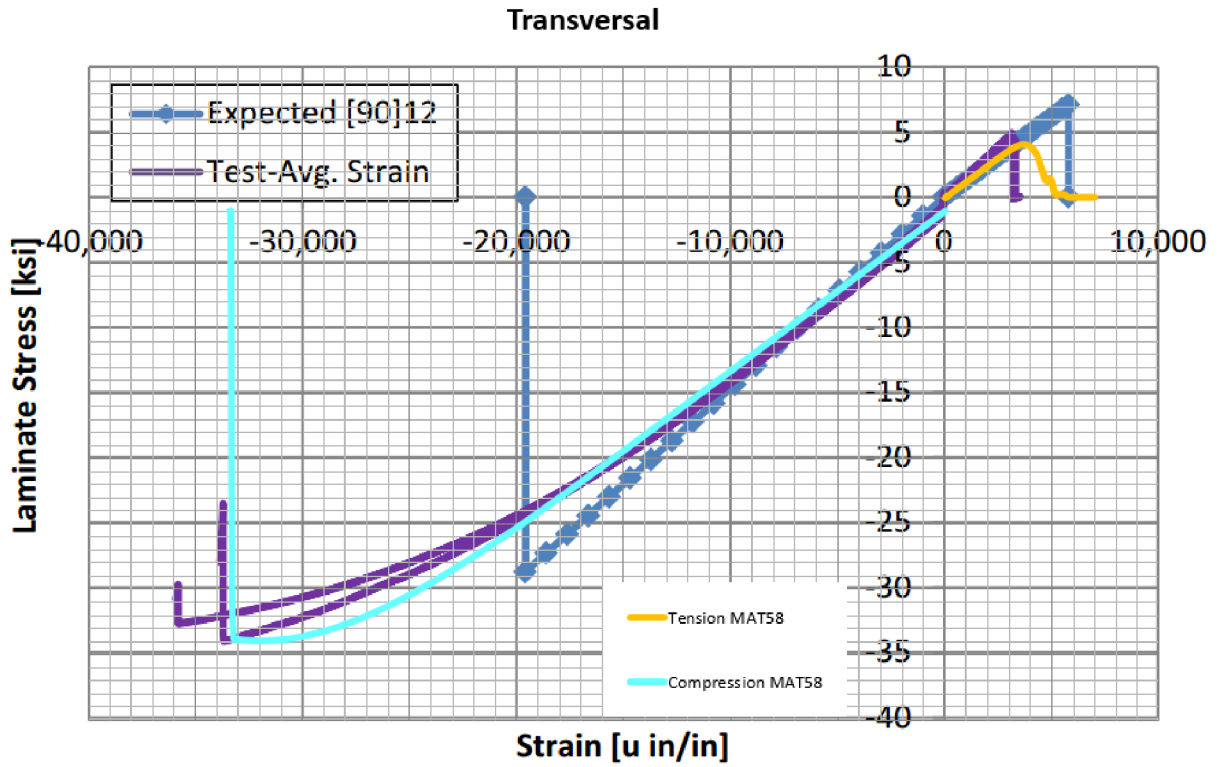


Figure 4 Stress vs. Strain – MAT58, transversal

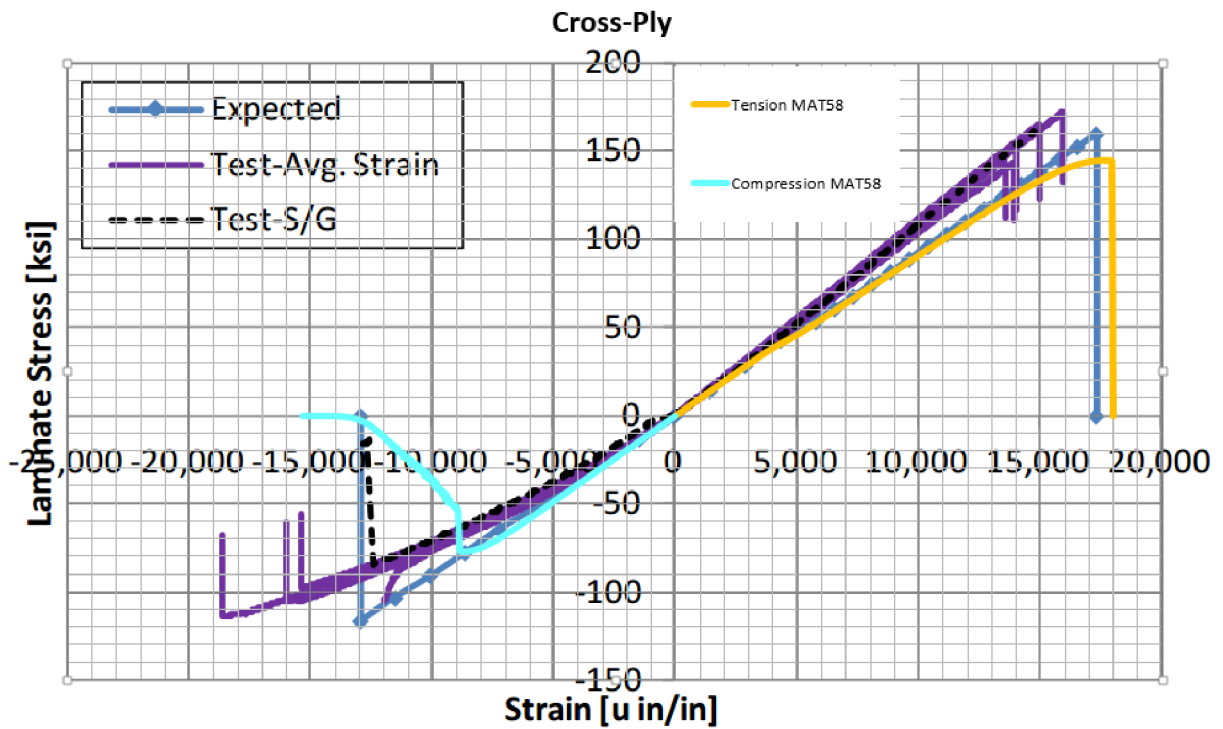


Figure 5 Stress vs. Strain – MAT58, cross-ply



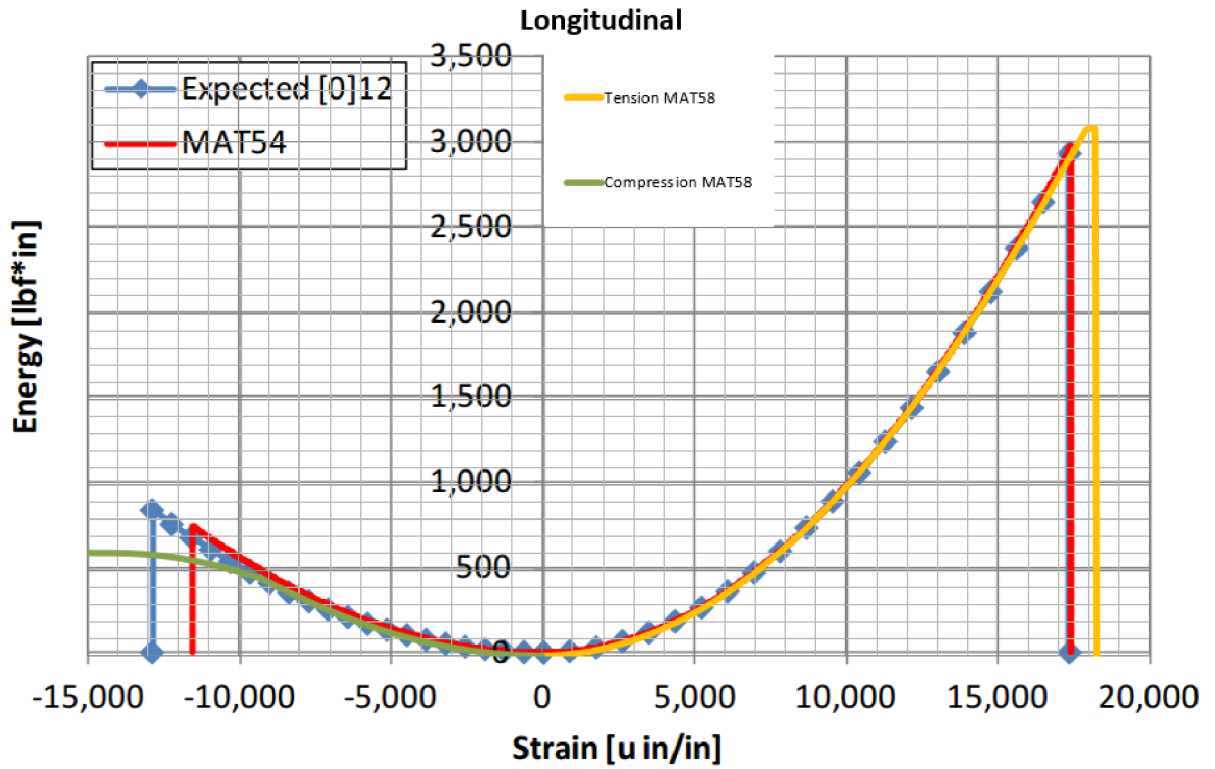


Figure 6 Energy vs. Strain – MAT58, longitudinal

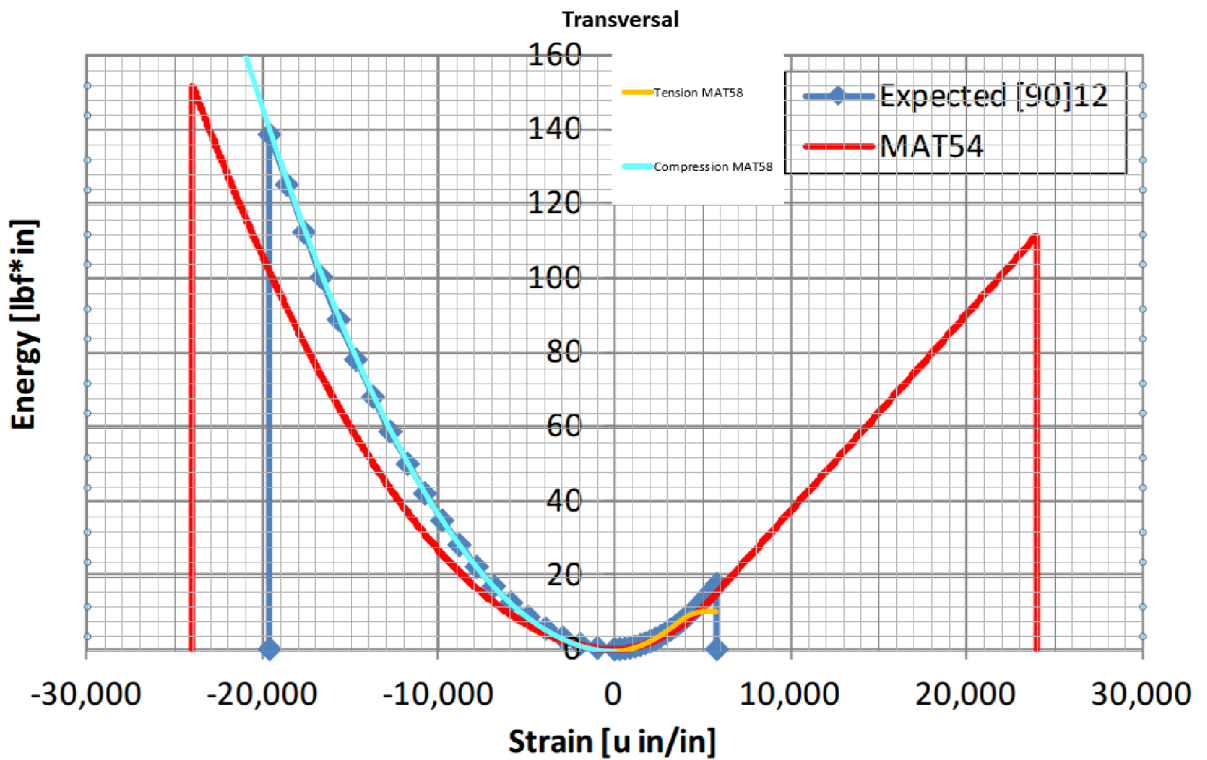
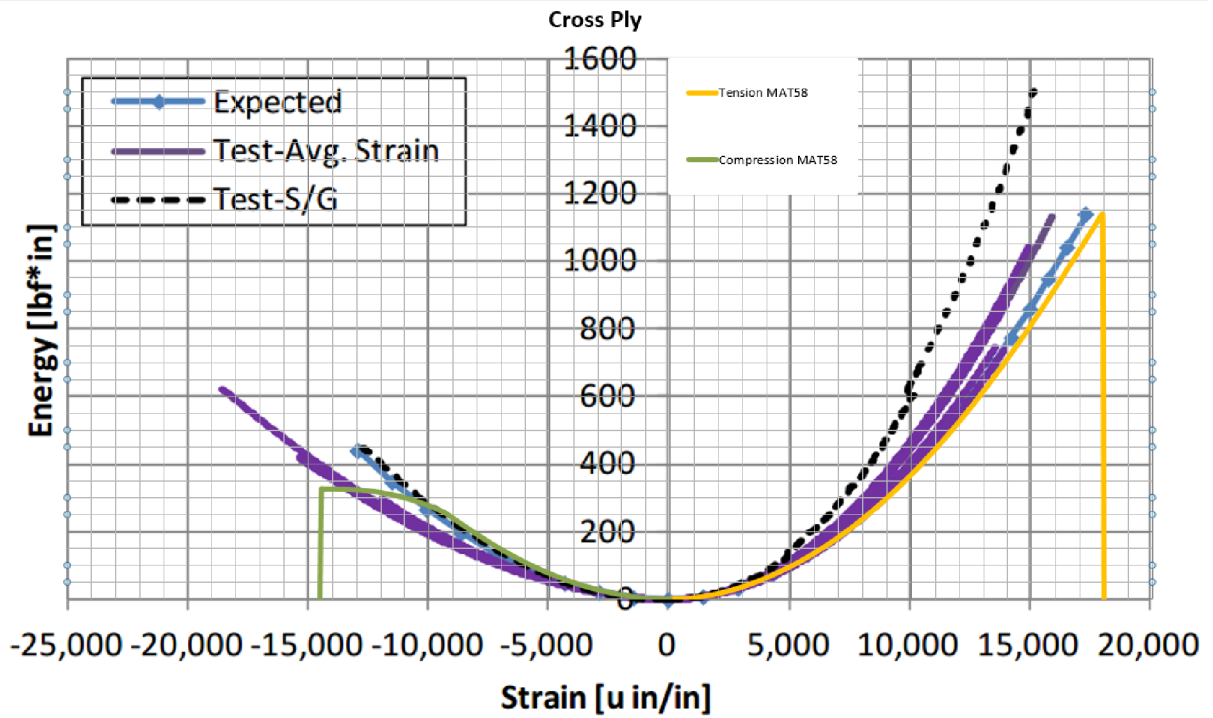


Figure 7 Energy vs. Strain – MAT58, transversal



**Figure 8 Energy vs. Strain – MAT58, cross-ply**

Tables 2, 3, 4 provide comparison of elastic modulus, failure strength, and peak energy between the experimental, theoretical (expected) and MAT58 results.

Loading	Quantity	Modulus[Msi]	Strength[ksi]	Energy[lbf*in]
Tension	Expected	18.1	314	2929
	MAT58	18.10	287.17	3080.28
	Error	0%	-9%	5%
	Test	18.84	309.8	2664
	MAT58	18.10	287.17	3080.28
	Error	-4%	-7%	16%
Compression	Expected	16.5	-213	834
	MAT58	18.10	-150.11	602.31
	Error	10%	-30%	-28%
	Test	16.29	-143	492
	MAT58	18.10	-150.11	602.31
	Error	11%	5%	22%

**Table 2 UD [0]<sub>12</sub> results - MAT58**



Loading	Quantity	Modulus[Msi]	Strength[ksi]	Energy[lbf*in]
Tension	Expected	1.22	7.1	18
	MAT58	1.22	4.06	10.93
	Error	0%	-43%	-39%
	Test	1.36	4.5	5
	MAT58	1.22	4.06	10.93
	Error	-10%	-10%	119%
Compression	Expected	1.47	-29	139
	MAT58	1.22	-33.00	381.92
	Error	-17%	14%	175%
	Test	1.57	-33	318
	MAT58	1.22	-33.00	381.92
	Error	-22%	0%	20%

**Table 3 UD [90]<sub>12</sub> results - MAT58**

Loading	Quantity	Modulus[Msi]	Strength[ksi]	Energy[lbf*in]
Tension	Expected	9.81	160	1140
	MAT58	9.66	144.40	1138.15
	Error	-2%	-10%	0%
	Test	10.58	157	1008
	MAT58	9.66	144.40	1138.15
	Error	-9%	-8%	13%
Compression	Expected	8.99	-116	441
	MAT58	9.66	-80.99	326.57
	Error	7%	-30%	-26%
	Test	8.84	-102	509
	MAT58	9.66	-80.99	326.57
	Error	9%	-21%	-36%

**Table 4 Cross-ply results - MAT58**

## 2.2.2 LS-DYNA MAT8 material model

Used simulation methods are the same as for MAT58 material model, so that boundary conditions and load are the same too.

### 2.2.2.1 Results

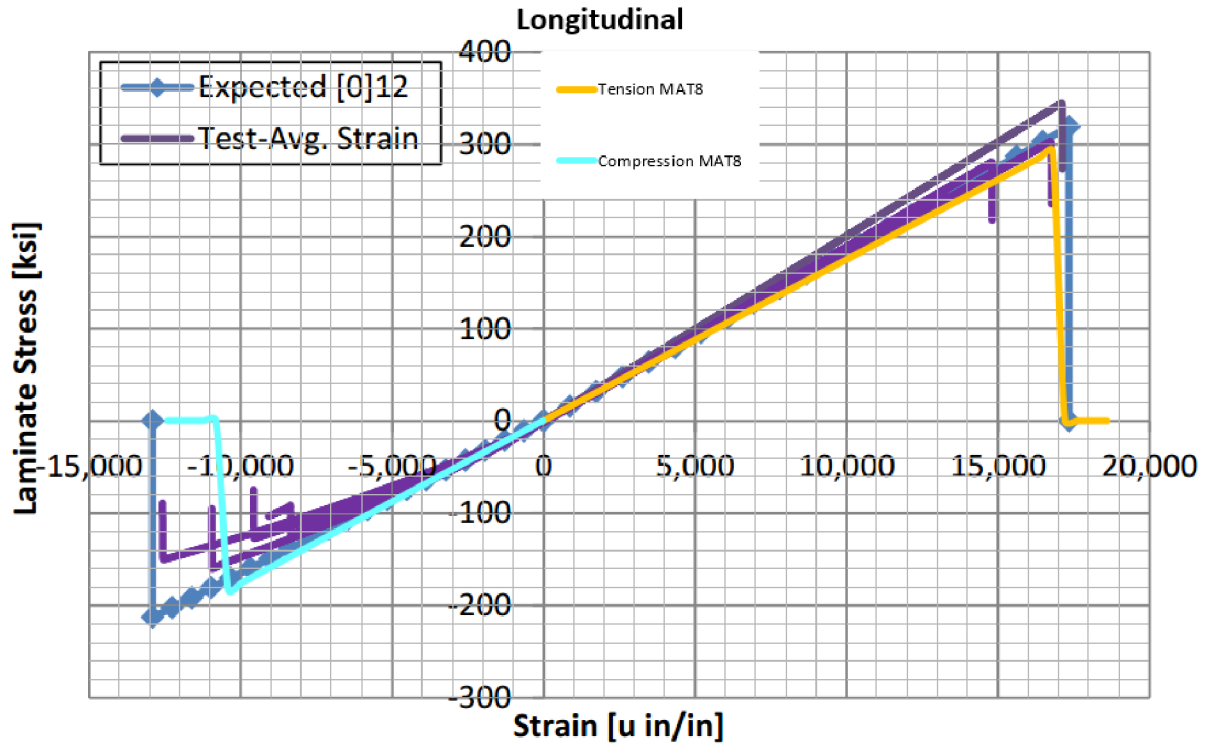


Figure 9 Stress vs. Strain – MAT8, longitudinal

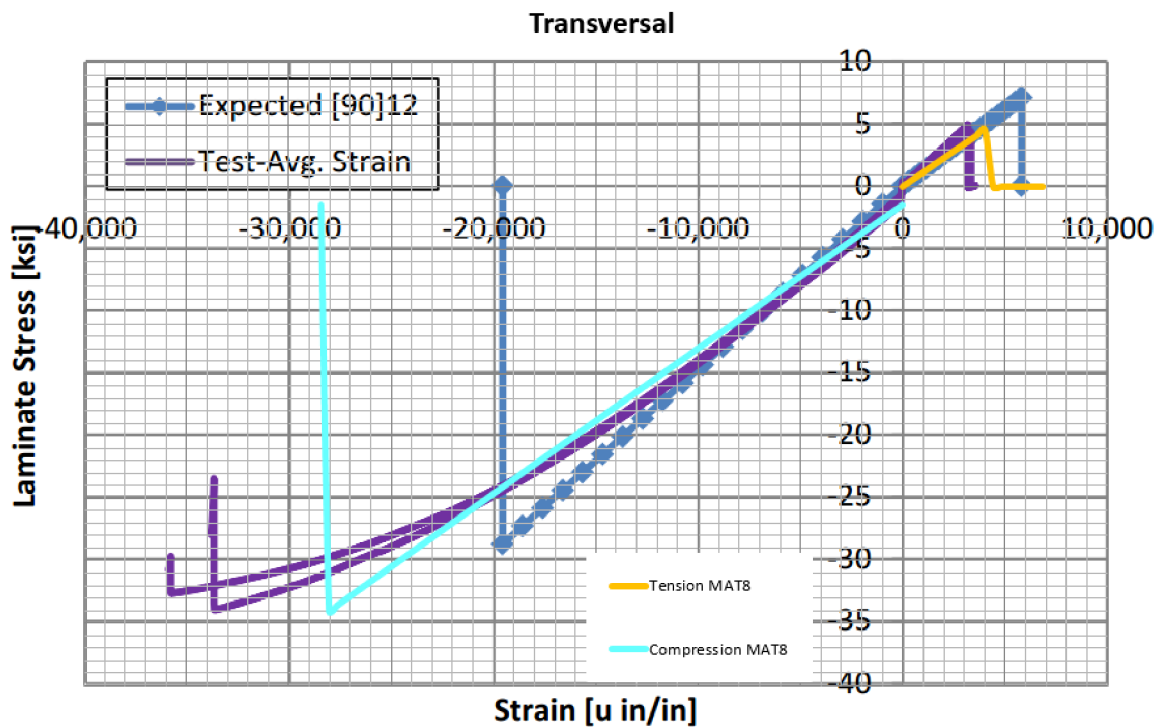


Figure 10 Stress vs. Strain – MAT8, transversal

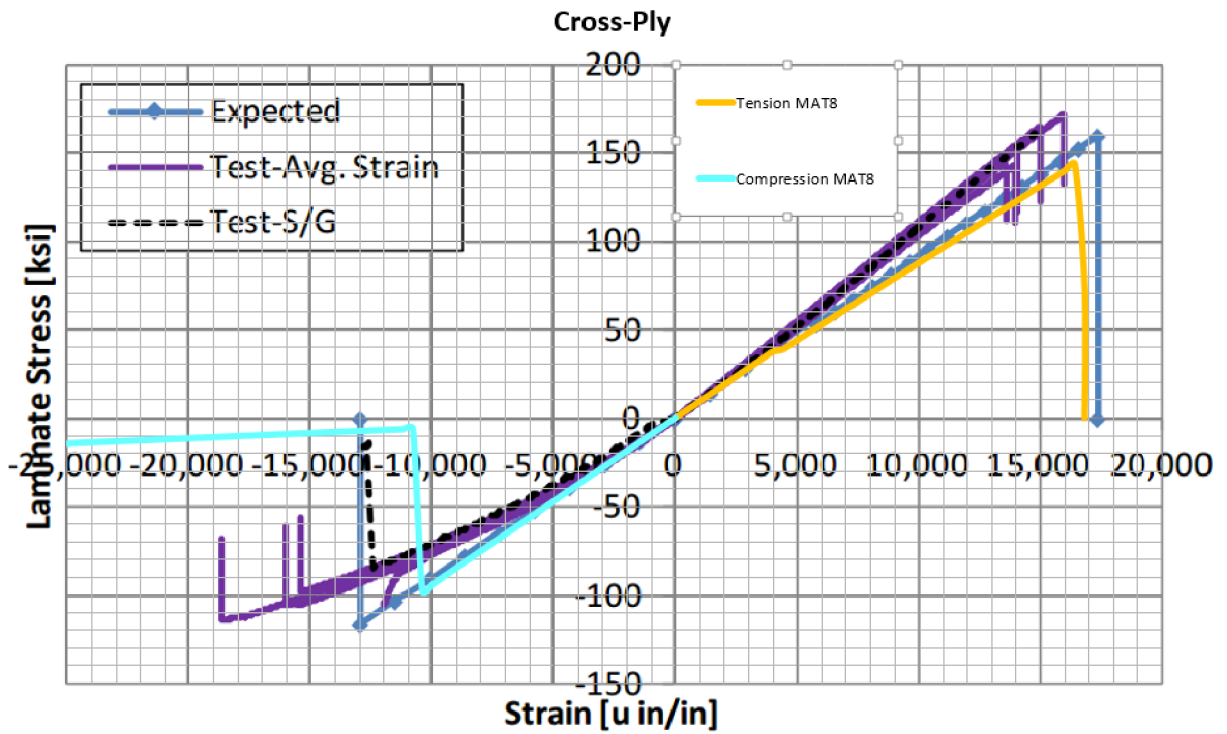


Figure 11 Stress vs. Strain – MAT8, cross-ply

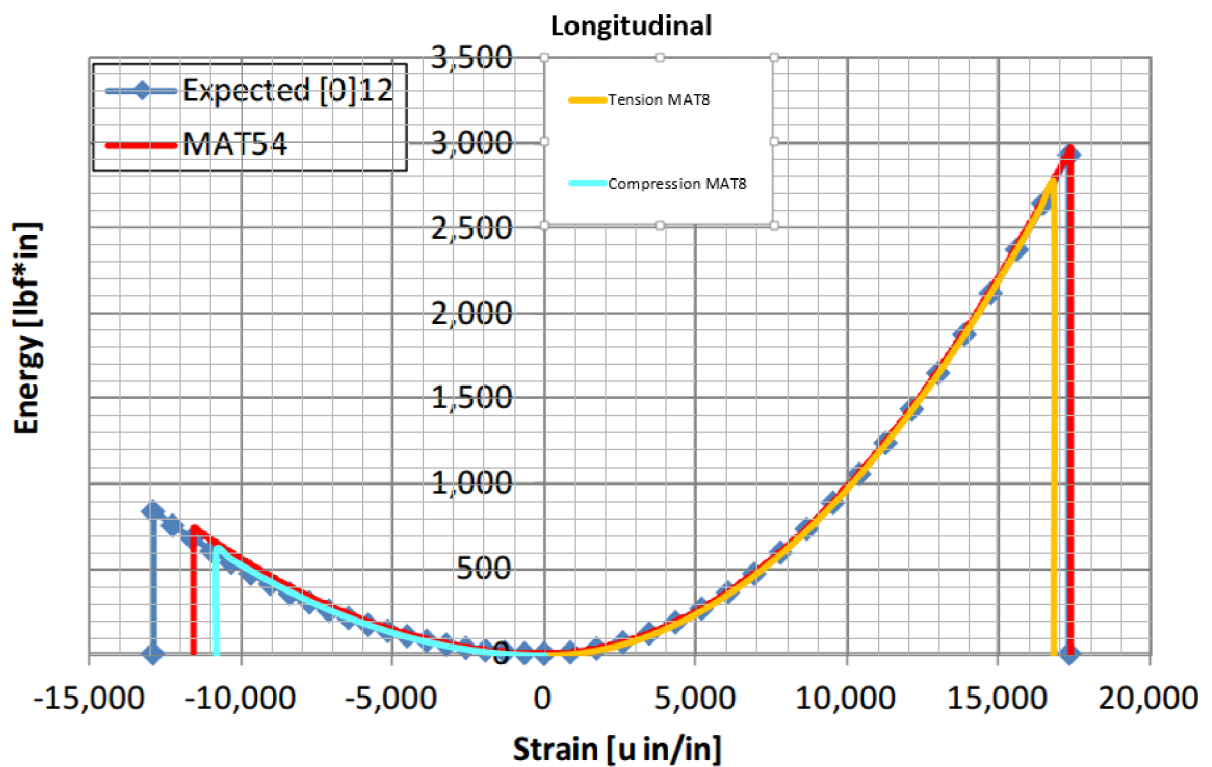


Figure 12 Energy vs. Strain – MAT8, longitudinal

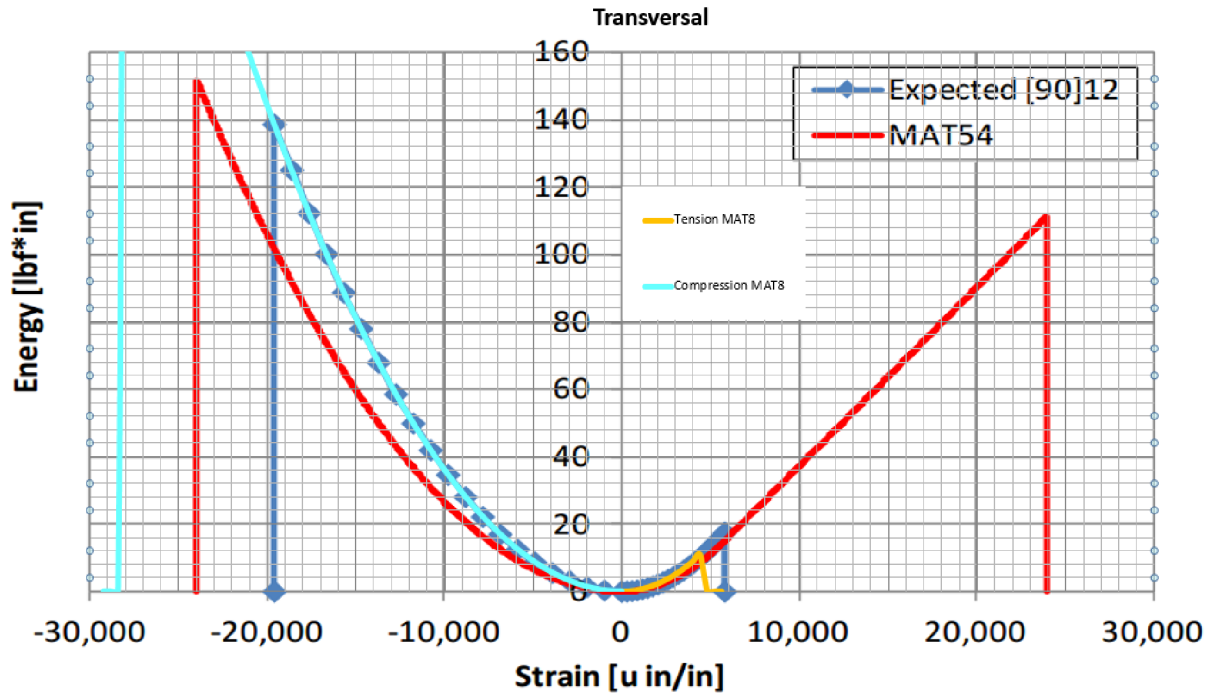


Figure 13 Energy vs. Strain – MAT8, transversal

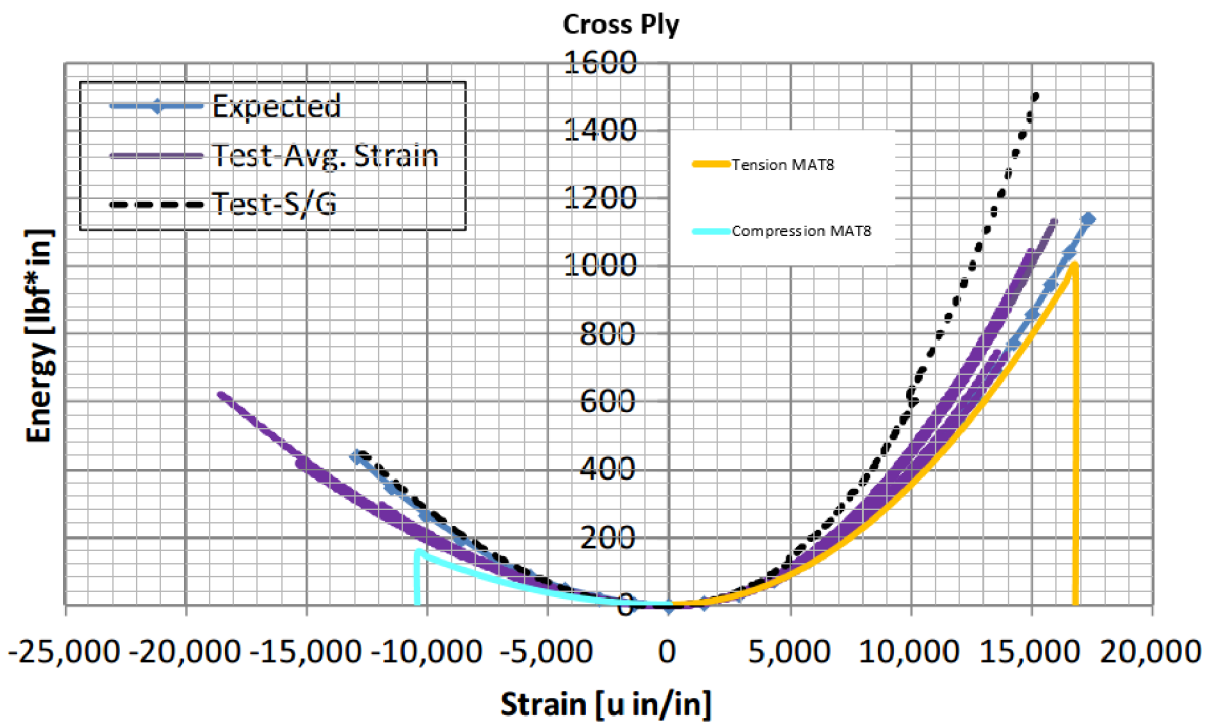


Figure 14 Energy vs. Strain – MAT8, cross-ply

Tables 5, 6, 7 provide comparison of elastic modulus, failure strength, and peak energy between the experimental, theoretical (expected) and MAT8 results.

Loading	Quantity	Modulus[Msi]	Strength[ksi]	Energy[lbf*in]
Tension	Expected	18.1	314	2929
	MAT8	17.55	292.66	2759.91
	Error	-3%	-7%	-6%
	Test	18.84	309.8	2664
	MAT8	17.55	292.66	2759.91
	Error	-7%	-6%	3%
Compression	Expected	16.5	-213	834
	MAT8	17.55	-183.63	619.85
	Error	6%	-16%	-35%
	Test	16.29	-143	492
	MAT8	17.55	-183.63	619.85
	Error	7%	22%	21%

**Table 5 UD [0]<sub>12</sub> results - MAT8**

Loading	Quantity	Modulus[Msi]	Strength[ksi]	Energy[lbf*in]
Tension	Expected	1.22	7.1	18
	MAT8	1.15	4.58	10.85
	Error	-6%	-55%	-66%
	Test	1.36	4.5	5
	MAT8	1.15	4.58	10.85
	Error	-19%	2%	54%
Compression	Expected	1.47	-29	139
	MAT8	1.15	-32.54	282.97
	Error	-28%	11%	51%
	Test	1.57	-33	318
	MAT8	1.15	-32.54	282.97
	Error	-37%	-1%	-12%

**Table 6 UD [90]<sub>12</sub> results - MAT8**

Loading	Quantity	Modulus[Msi]	Strength[ksi]	Energy[lbf*in]
Tension	Expected	9.81	160	1140
	MAT8	9.35	143.51	999.32
	Error	-5%	-11%	-14%
	Test	10.58	157	1008
	MAT8	9.35	143.51	999.32
	Error	-13%	-9%	-1%
Compression	Expected	8.99	-116	441
	MAT8	9.35	-98.28	151.52
	Error	4%	-18%	-191%
	Test	8.84	-102	509
	MAT8	9.35	-98.28	151.52
	Error	5%	-4%	-236%

**Table 7 Cross-ply results - MAT58**

## 2.3 Material models comparison

Comparison is based on an assessment of a summary of errors in longitudinal, transversal directions, and cross-ply case in relation to the experimental data. Results are shown below in tables 8-12.

Loading	Quantity	Strength[ksi]	Energy[lbf*in]
Tension	MAT54	3%	11%
	MAT58	7.30%	15.63%
	MAT8	5.53%	2.29%
Compression	MAT54	49%	51%
	MAT58	4.97%	22.42%
	MAT8	8.59%	9.76%

**Table 8 Summary of errors – longitudinal**

Loading	Quantity	Strength[ksi]	Energy[lbf*in]
Tension	MAT54	57%	220%
	MAT58	9.76%	118.61%
	MAT8	1.69%	73.12%
Compression	MAT54	79%	50%
	MAT58	0.01%	20.10%
	MAT8	1.38%	14.44%

**Table 9 Summary of errors – transversal**

Loading	Quantity	Strength[ksi]	Energy[lbf*in]
Tension	MAT54	2%	11%
	MAT58	8.73%	11.44%
	MAT8	8.61%	3.78%
Compression	MAT54	4%	4%
	MAT58	25.94%	55.86%
	MAT8	18.57%	53.28%

**Table 10 Summary of errors – cross ply**

Loading	Quantity	Strength[ksi]	Energy[lbf*in]
Tension	MAT54	21%	81%
	MAT58	8.60%	48.56%
	MAT8	5.28%	26.40%
Compression	MAT54	44%	35%
	MAT58	10.31%	32.79%
	MAT8	9.51%	25.83%

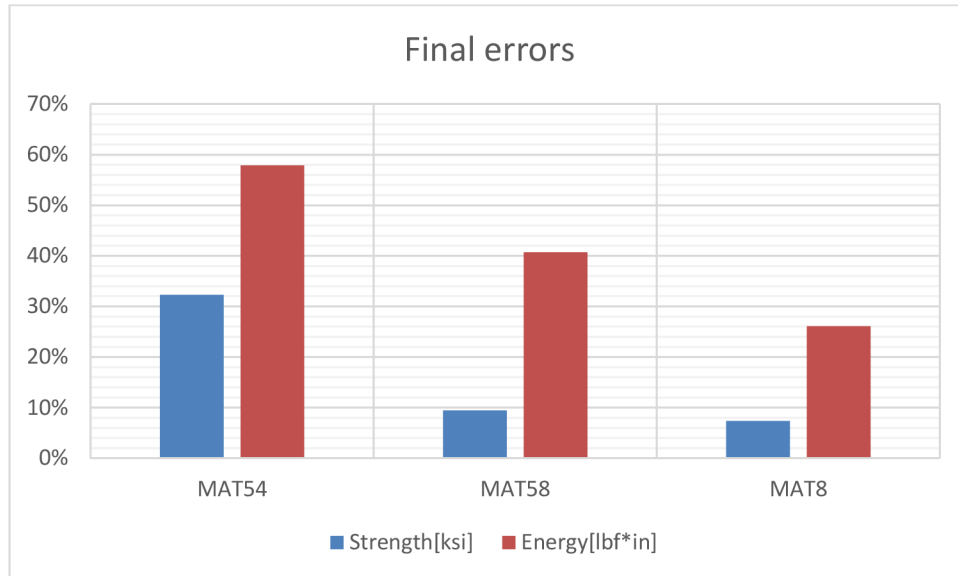
**Table 11 Summary of errors – mean**

Quantity	Strength[ksi]	Energy[lbf*in]
MAT54	32%	58%
MAT58	9.45%	40.68%
MAT8	7.39%	26.11%

**Table 12 Summary of errors – final**

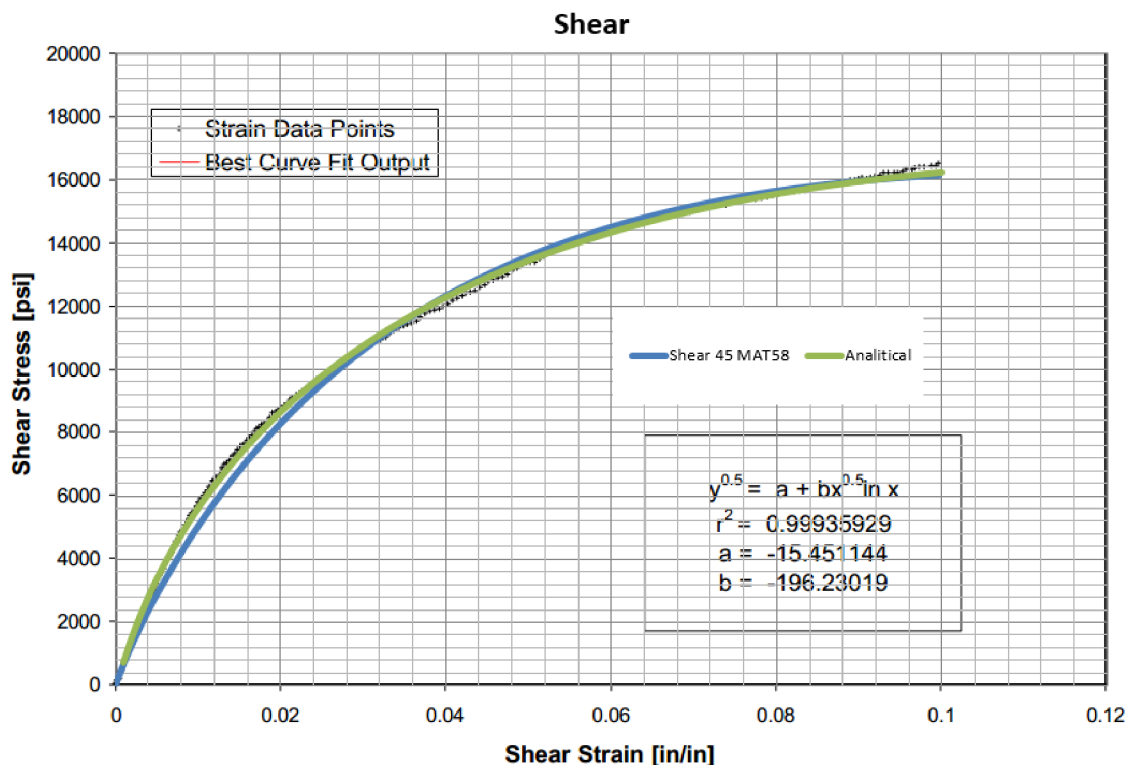
## 2.4 Material model choosing

Final error was calculated as average value from tension and compression (longitudinal, transversal, cross-ply). As shown on Figure 15 the least computational error has MAT8 material model, so that it was chosen for further bird strike simulation.



**Figure 15 Final errors – comparison**

MAT8 is simpler than MAT58 material model, which is the main advantage of this model. However, this material model can only simulate linear shear stress-strain curve, which is the main disadvantage in comparison to MAT58 material model. Behavior of these models at shear loading is shown below.



**Figure 16 Stress vs. Strain – MAT58, tensile test of a  $\pm 45^\circ$  laminate**

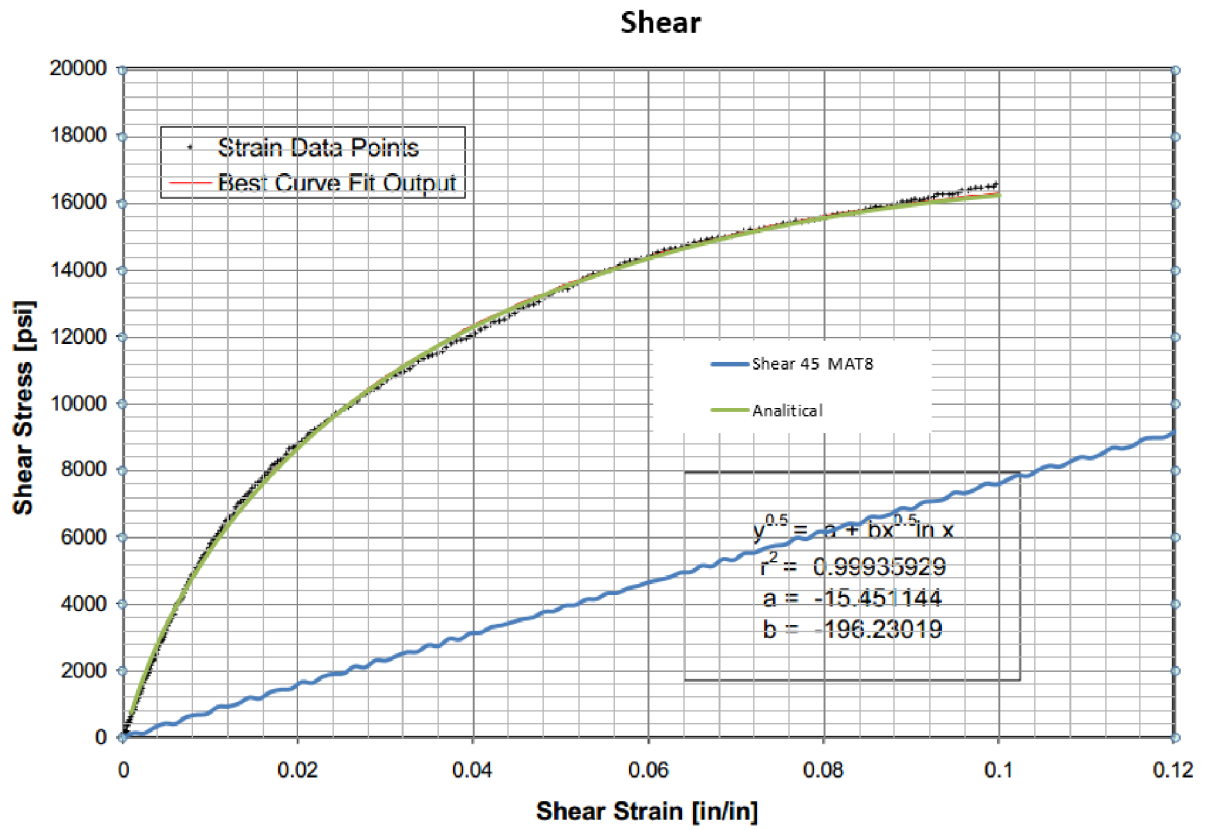


Figure 17 Stress vs. Strain – MAT8, tensile test of a  $\pm 45^\circ$  laminate

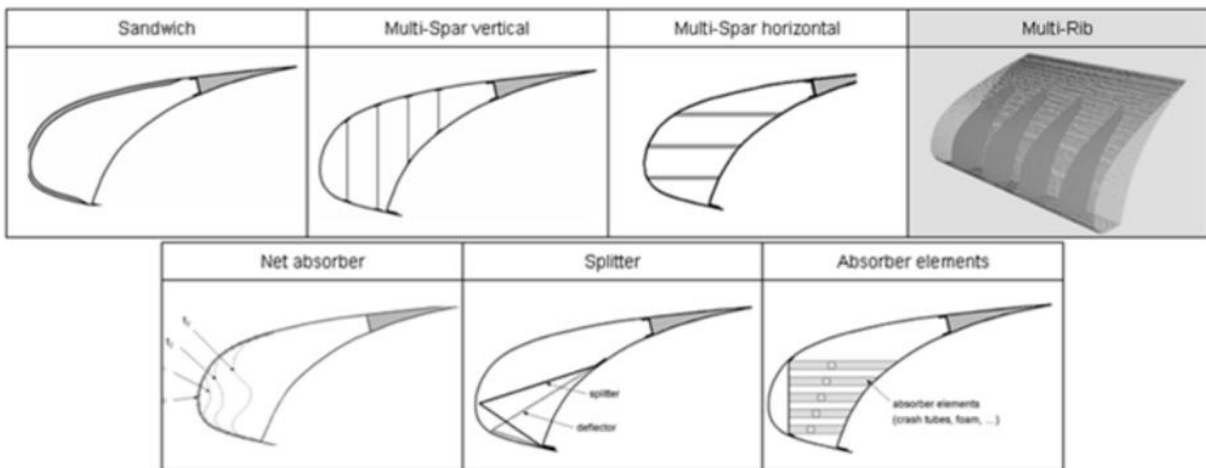


### 3 Slat construction

Slats are aerodynamic surfaces on the leading edge of the wings which, when deployed, allow the wing to operate at a higher angle of attack. A higher coefficient of lift is produced as a result of angle of attack and speed, so by deploying slats an aircraft can fly at slower speeds, or take off and land in shorter distances. They are usually used while landing or performing maneuvers which take the aircraft close to the stall, but are usually retracted in normal flight to minimize drag.

#### 3.1 Slat concept

In order to make an efficient slat it is needed to count mutual influences of several factors, such as manufacturing, impact performance, weight and costs. According to [7] there are number of concepts available such as sandwich, multi-spar vertical, multi-spar horizontal, multi-rib, net absorber, splitter, absorber elements concept. Schematically they are shown below.



**Figure 18** Various concepts for composite slat design

Because of showing best potential of energy absorption capacity [7], the Multi-Rib concept was selected for further analysis.

#### 3.2 Airplane prototype

Because of availability of technical and construction data **Boeing B737-200** was chosen as an airplane prototype. Wing geometry and FE models have been created based on information from technical manuals.

The Boeing 737-200 is a twin-engine short-range narrow body airliner with a capacity of maximum 136 passengers produced by the American manufacturer *Boeing Commercial Airplanes*.

### 3.3 Slat location

Designing slat is located on an outer part of the wing. Its location is shown below.

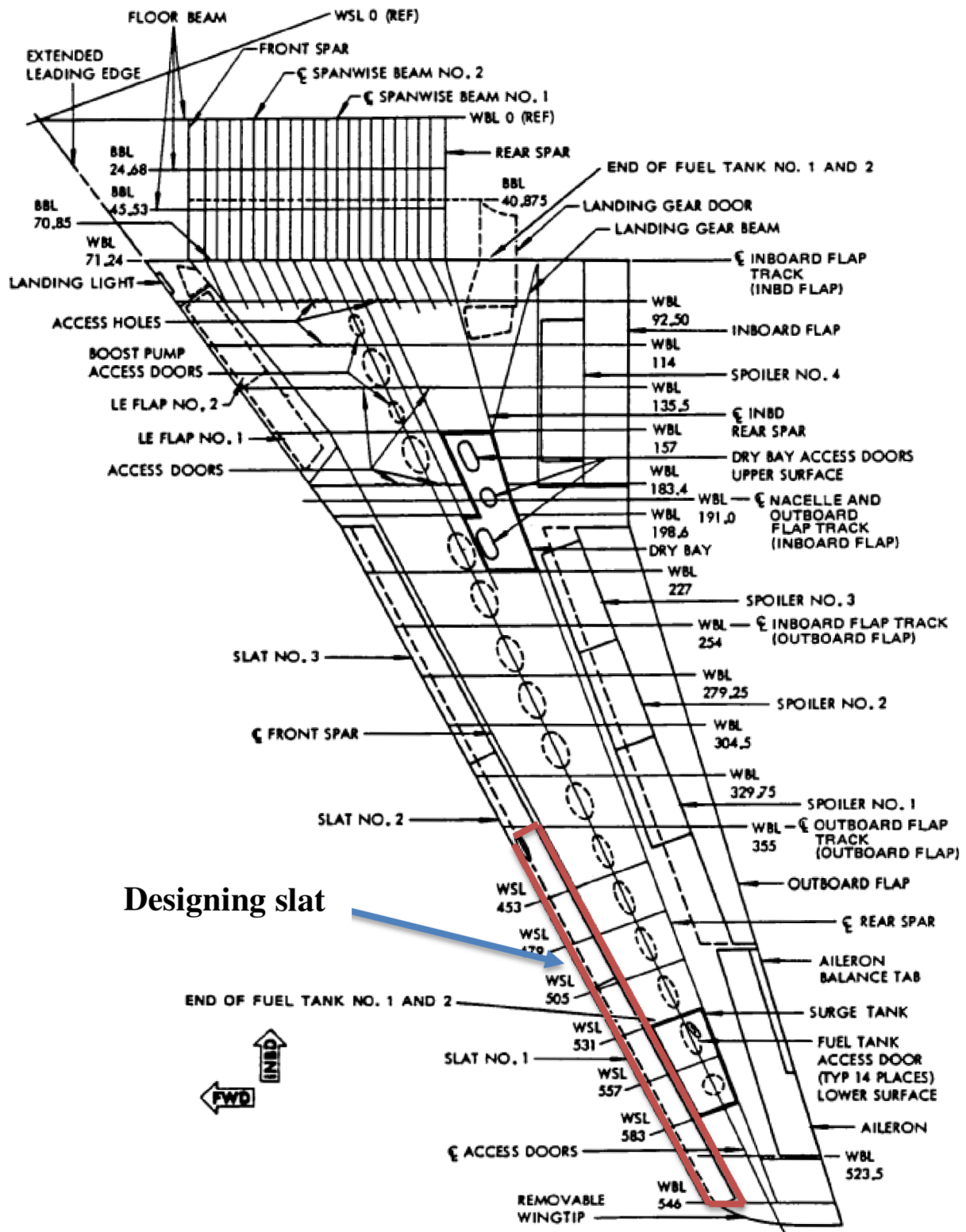


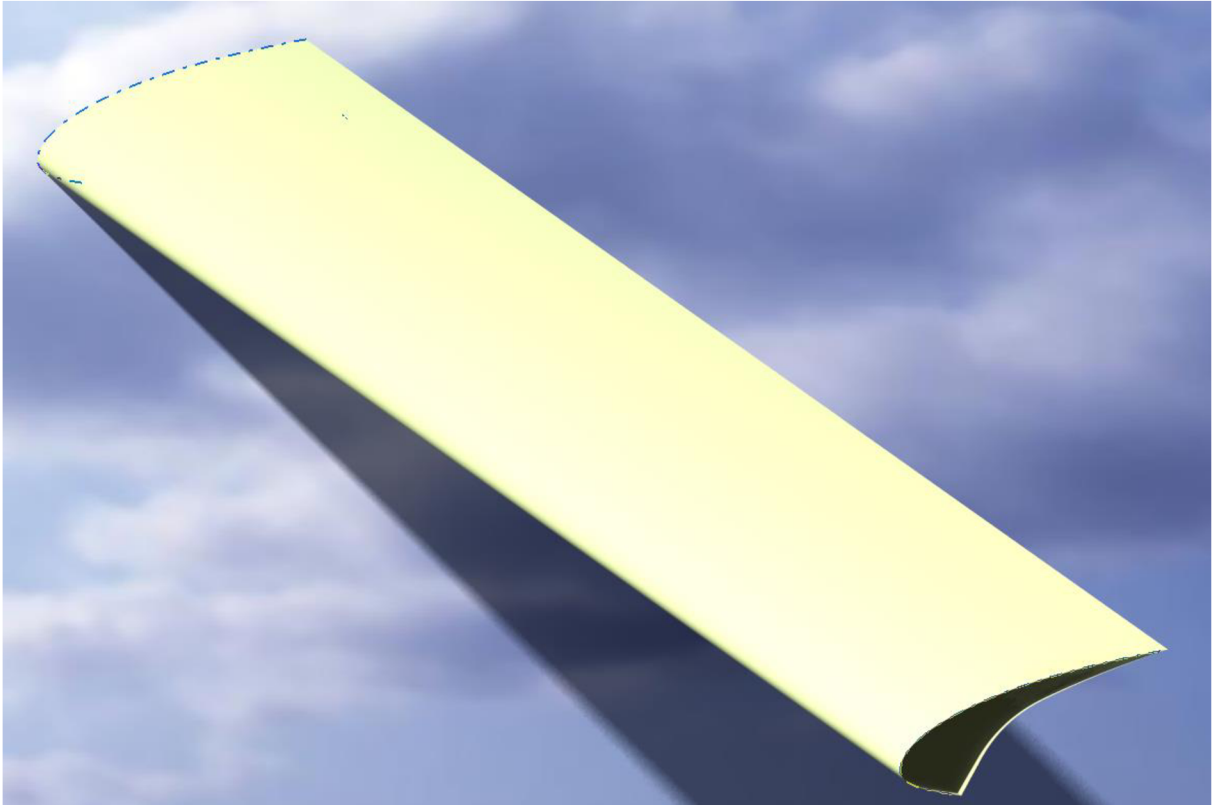
Figure 19 Slat location

### 3.4 Slat geometry

Original construction has three slats. In this project, these three slats will be substituted with two longer slats. Taking into account the fact that the end part of the wing has the least resistance to a bird strike (significantly smaller stiffness in compare to root section) the outer slat will be designed in this project.

#### 3.4.1 CAD model

Airfoil geometry was taken from an earlier project [6]. CAD model was created in CATIA V5 software.



**Figure 20 Slat CAD model**

Geometric characteristics: length is 4773 mm, cross-section area is 4275,91 mm<sup>2</sup>.

### 3.5 Slat attachment position

Attachment position was taken from [8] and shown below. In local coordinate system coordinates are:  $x = 30.75\text{mm}$ ,  $y = -3.85\text{mm}$

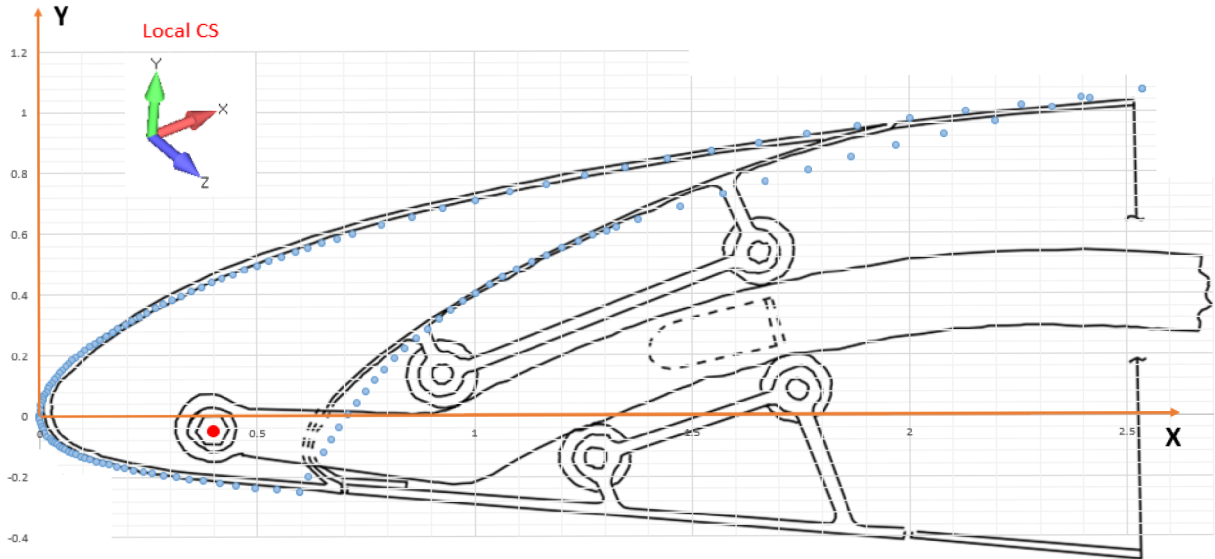


Figure 21 Attachment position in local CS

### 3.6 Material

IM7/8552 UD tape was chosen [9] for project purposes. The material properties are tabulated in Table 13 below.

Material props		
E1t	158830	MPa
E1c	143700	MPa
E2t	9100	MPa
E2c	9700	MPa
$\mu_{12}$	0.36	
$\mu_{21}$	0.02063	
G12s	4800	MPa
F1tu	2096	MPa
F1cu	1126	MPa
F2tu	81	MPa
F2cu	223	MPa
F12su	134	MPa
t0	0.125	mm

Table 13 IM7/8552 material properties

### 3.7 Slat loads

During the flight slat is loaded by aerodynamic forces, which are the main input for further calculations. Because of limited information accessible about the prototype only three load cases (take-off, cruise, landing) were chosen for further calculations. The maximum loads appeared to be in the take-off configuration [6], which parameters are shown below.

#### 1.2 Take off configuration #5

$m=62822\text{kg}$  (=MTOW)

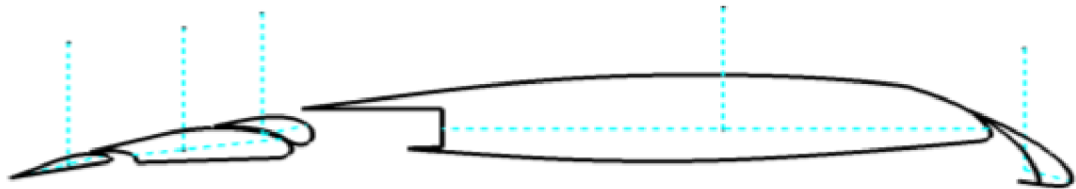
$M=0,34$  (max speed with flaps #5)

$H=0\text{m}$

$n=2$  (flap envelope)

$v_{IAS}=115,6\text{m/s}$  (=416km/h=224,7KIAS)

Místo	Slot	Dutina	Flap 1	Flap 2	Flap 3	Celkem
$\Delta\text{CL}[-]$	0,70	1,99	0,11	0,01	0,05	2,86
$\Delta\text{CL}[\%]$	24,48	69,58	3,85	0,35	1,75	100



Místo	Slot	Dutina	Celkem
$\Delta\text{CL}[-]$	0,70	2,16	2,86
$\Delta\text{CL}[\%]$	24,48	75,52	100

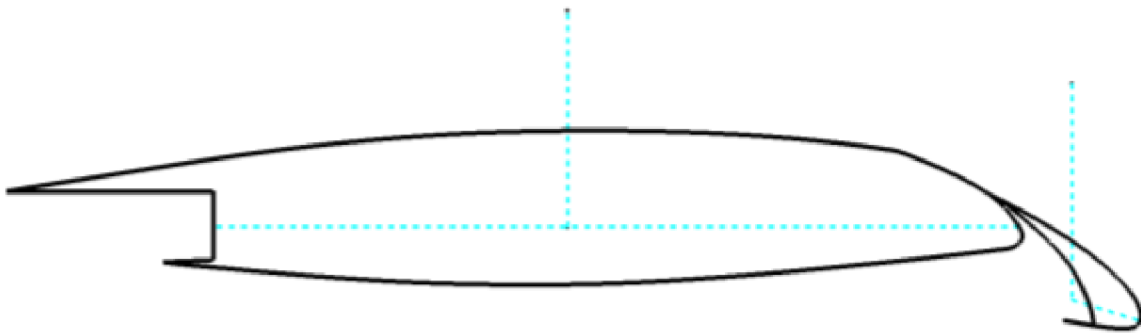


Figure 22 Take off configuration parameters

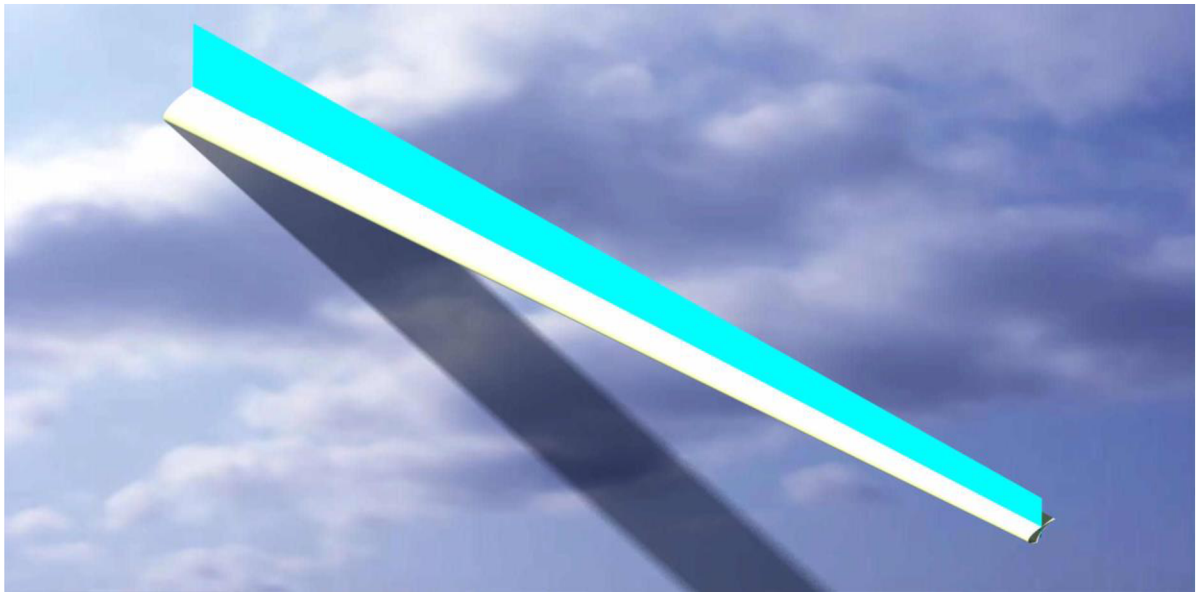
#### 3.7.1 Analytical loadings calculation

Continuous load  $q$  taken from [6]. Safety factor is 1.725.

Limit loads[N/mm]	
q1	1.735
q2	5.839

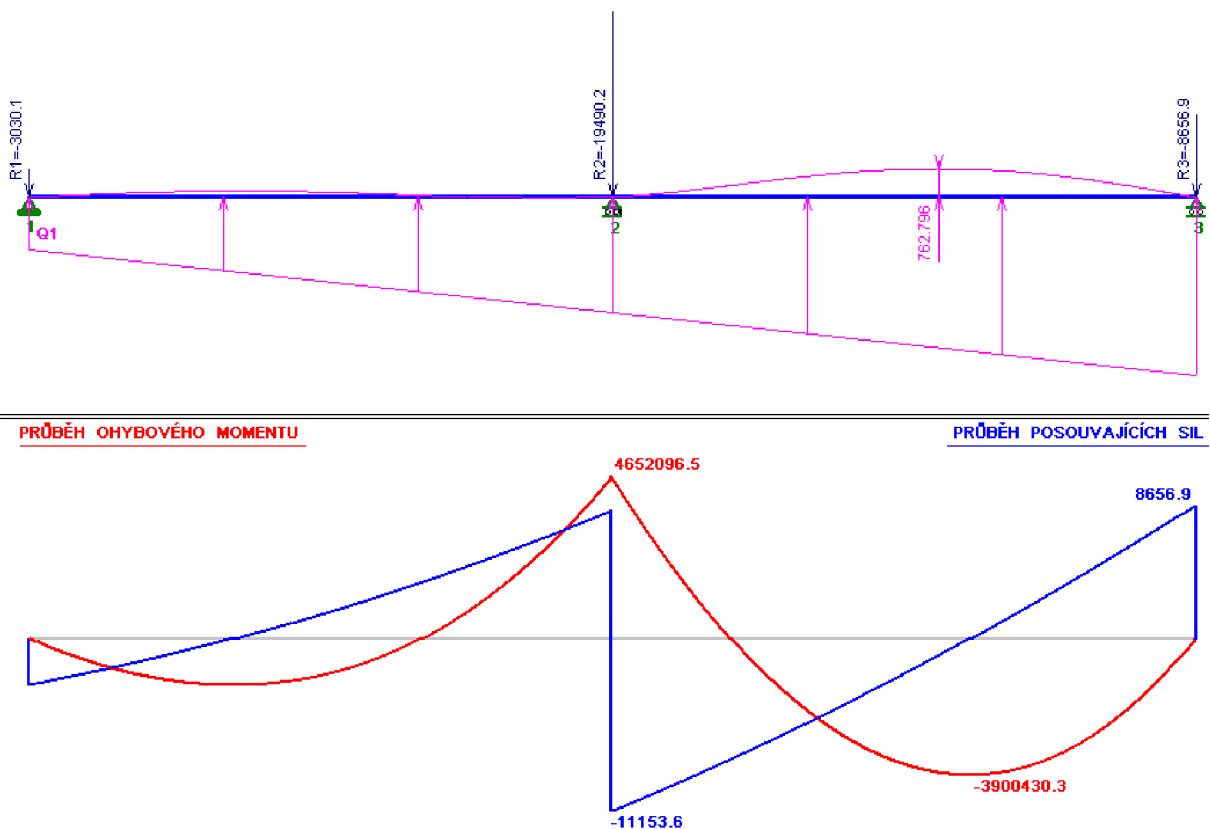
Ultimate loads[N/mm]	
q1	2.993
q2	10.072

Table 14 Beam continuous loading



**Figure 23 Wing continuous loading**

During static analysis slat considered as a beam supported at three points. Internal load factors distribution along the slat was calculated in MINISTATIC software. The results are shown below.



**Figure 24 Beam static analysis**

The maximum torsion moment (occurred close to the mid support) can be calculated from the next equation:

$$R \cdot a - T \cdot b = 0 \quad (1)$$

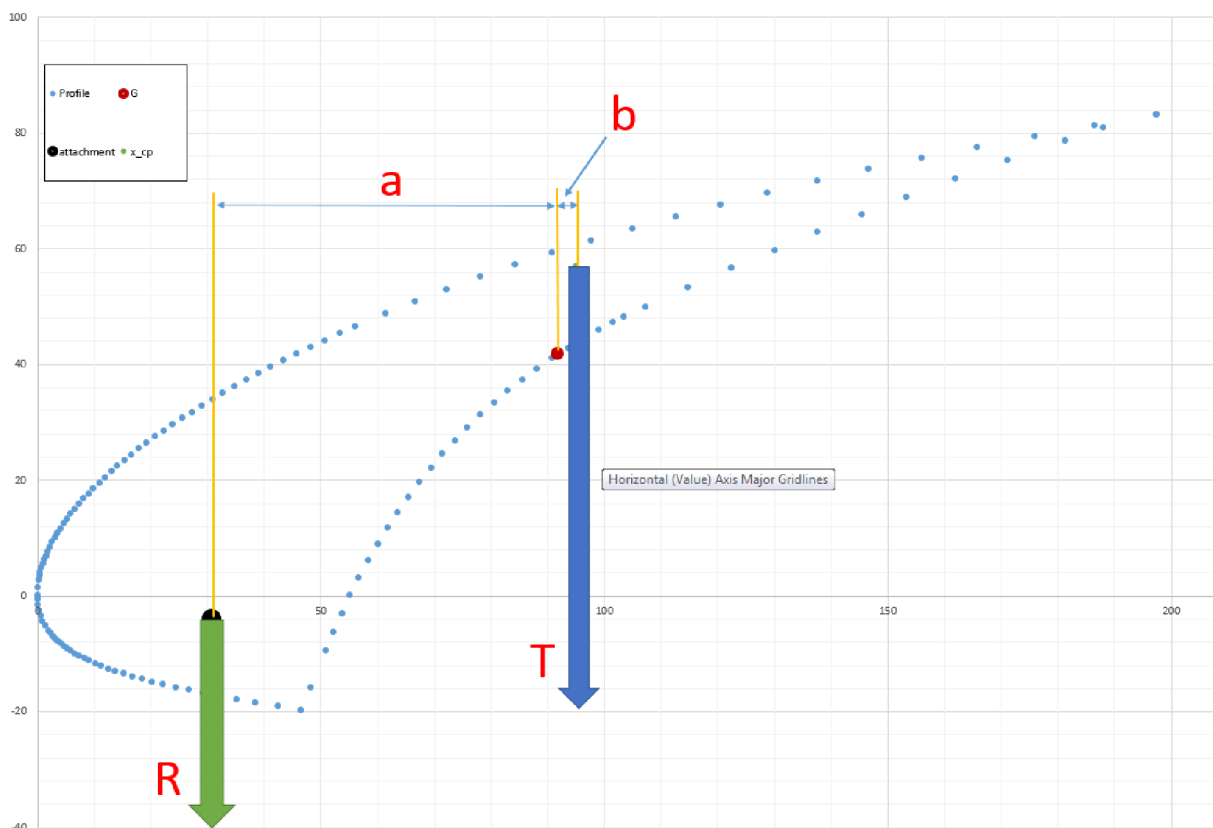
where:

- $R=19490$  N – reaction at the middle support
- $T=11154$  N – the maximum shear force at the middle support
- $a=60.93$ mm,  $b=3.4$ mm – forces arms

Note that the moment is calculated in relation to the elastic axis of the slat cross section. Schematically it is shown below on Figure 23.

Loading factor	Value
Shear force [N]	11154
Bending moment [N*mm]	4652097
Torsion moment [N*mm]	-1149780

**Table 15 Beam loads**



**Figure 25 Slat torsion moment**



### 3.8 Slat thickness calculation

The skin is planned to be made from a laminate with symmetrical balanced layup. According to the differential principle for composite thin-walled structure development the torsion moment and shear force are supposed to be taken by plies with  $\pm 45^\circ$  orientation and bending moment by plies with  $0^\circ$  orientation. Because none forces act in transversal direction (-X in local CS) laminate has  $\pm 45^\circ/0^\circ$  layup.

#### 3.8.1 Calculation methodology

Cross-sectional internal loads and stresses were calculated according to methodology [10]. The slat is considered as a thin-walled rod with a closed loop section with cross-section constant along the slat length.

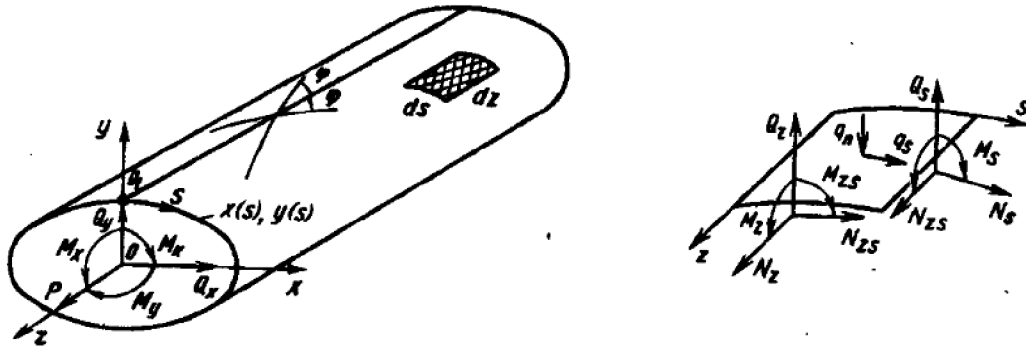


Figure 26 Composite rod geometrical parameters and loads

Torsion moment and shear force were calculated to the shear stresses flow  $\mathbf{q}$  and bending moment to longitudinal forces  $\mathbf{p}$ .

List of used equations is provided below.

Longitudinal forces:

$$p = B(s) \left[ \frac{P}{S} + k \left( \frac{\overline{M_x}}{D_x^0} \bar{y} + \frac{\overline{M_y}}{D_y^0} \bar{x} \right) \right] \quad (2)$$

where  $\bar{x} = x - x_0 - n_x(y - y_0)$ ,  $\bar{y} = y - y_0 - n_y(x - x_0)$ ,

$$k = \frac{1}{1 - n_x n_y}, n_x = \frac{D_{xy}^0}{D_x^0}, n_y = \frac{D_{xy}^0}{D_y^0}, \overline{M_x} = M_x - y_0 P, \overline{M_y} = M_y - x_0 P \quad (3)$$

$$D_x^0 = D_x - y_0^2 S, D_y^0 = D_y - x_0^2 S, D_{xy}^0 = D_{xy} - x_0 y_0 S \quad (4)$$

$$x_0 = \frac{S_y}{S}, y_0 = \frac{S_x}{S}, S = \oint B ds, S_x = \oint B y ds, S_y = \oint B x ds,$$

$$D_x = \oint B y^2 ds, D_y = \oint B x^2 ds, D_{xy} = \oint B xy ds \quad (5)$$

Where  $S$  is the elastic slat cross-section modulus,  $S_x$ ,  $S_y$  are first moments of the slat cross-section area,  $D$  are inertia moments,



$B = E_x$ - laminate Young's modulus in longitudinal direction

Because moment over y-axis  $M_y = 0$  and axis force  $P = 0$ , equation (2) reduces to:

$$p = B(s) \cdot k \frac{\overline{M_x}}{D_x^0} \overline{y} \quad (6)$$

Shear flow:

$$q = q_0 + q_Q \quad (7)$$

where

$$q_Q = -k \left[ \frac{Q_y}{D_x^0} \overline{S_x}(s) + \frac{Q_x}{D_y^0} \overline{S_y}(s) \right] \quad (8)$$

$$\overline{S_x}(s) = \int_0^s B \overline{y} ds, \overline{S_y}(s) = \int_0^s B \overline{x} ds \quad (9)$$

Where functions  $\overline{S_x}(s), \overline{S_y}(s)$  are static moments of the cut part of the cross-section envelope at s-coordinate to x and y axis (see Figure 26), and  $Q_y, Q_x$  are shear forces

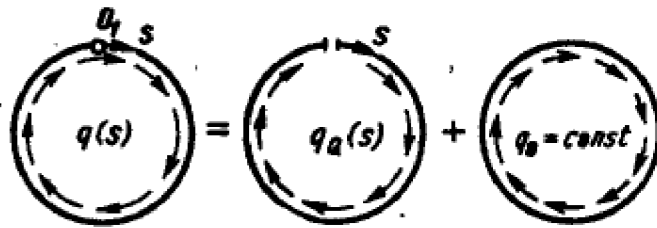


Figure 27 Shear stress flow

$$q_0 = \frac{1}{2F} (M_t + \oint q_Q r ds) \quad (10)$$

$$r = x \sin \beta + y \cos \beta \quad (11)$$

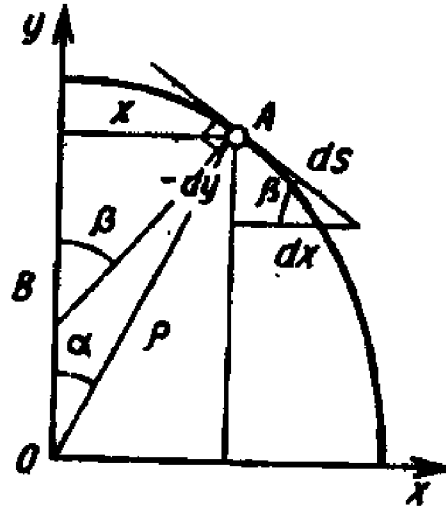


Figure 28 Composite rod geometrical parameters

Because  $Q_x = 0$ , equation (6) reduces to:

$$q = -k \frac{Q_y}{D_x^0} \bar{S}_x(s) + \frac{1}{2F} (M_t + \oint q_Q r ds) \quad (12)$$

Laminate properties were calculated according to methodology [11]

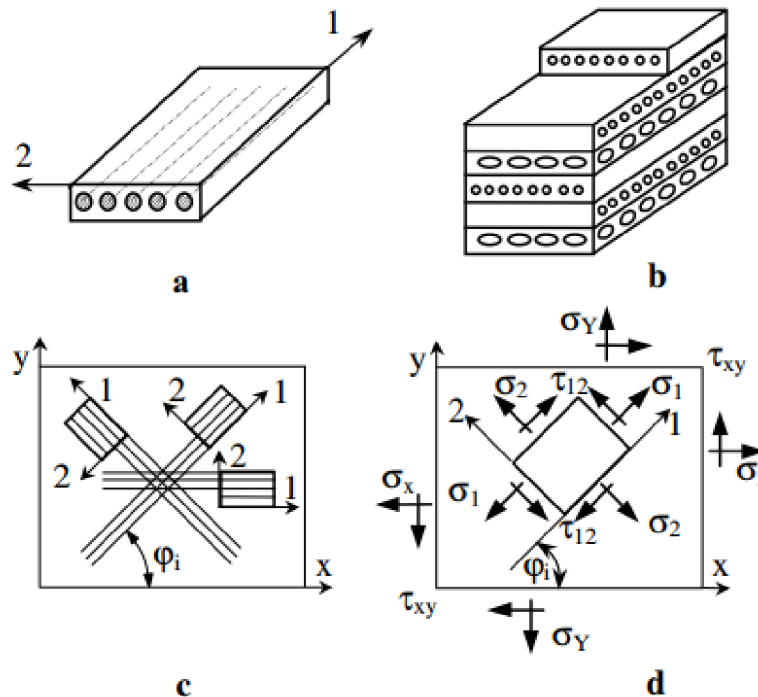


Figure 29 Formulation of composite properties (according to [11])

Formulas for determination of elastic properties of laminated composite materials follow from [11].

### 3.8.2 Calculation algorithm

Calculation done by the next iteration sequence:

1. Random number of  $\pm 45^\circ/0^\circ$  layers was set.
2. Stresses from shear stresses flow  $\mathbf{q}$  and longitudinal forces  $\mathbf{p}$  were calculated separately for plies in corresponding directions ( $\pm 45^\circ$  from  $\mathbf{q}$ ,  $0^\circ$  from  $\mathbf{p}$ ).
3. Reserve factor was calculated, and depend on its value the number of plies was corrected separately for corresponding reinforcement angle
4. Laminate properties (elastic constants) were determined
5. Laminate stresses  $\sigma_x, \tau_{xy}$  were determined
6. Ply stresses were calculated for each ply angle ( $\pm 45^\circ$  and  $0^\circ$ )
7. Reserve factor was determined, and depend on its value the number of plies was corrected

Note, that target RF=1.10, critical stress is in compression.

### 3.8.3 Calculation results

Calculation in details can be found in supported document [12]. Resultant laminate properties, stresses, ply stresses and MSs are shown below.

$\rho$ [N/mm]	$q$ [N/mm]
-1616.88	-1851.45

**Table 16 Internal cross-sections I load factors**

Elastic constants											
$E1_i$ [MPa]	$E2_i$ [MPa]	$n$ [-]	$\delta$ [mm]	$\phi$ [deg]	$\cos\phi$	$\sin\phi$	$\cos 2\phi$	$\sin 2\phi$			
160018.183	9168.08	15	0.125	45	0.707	0.707	0	1			
$b_{11i}$	$b_{12i}=b_{21i}$	$b_{22i}$	$b_{13i}=b_{31i}$	$b_{23i}=b_{32i}$	$b_{33i}$	B11	B12	B22	B13	B23	B33
48746.8182	39146.82	48746.82	37712.53	37712.53	40646.31	91400.28	73400.284	91400.28	70710.99	70710.99	76211.83

**Table 17 Plies elastic constants,  $45^\circ$  plies**

Elastic constants											
$E1_i$ [MPa]	$E2_i$ [MPa]	$n$ [-]	$\delta$ [mm]	$\phi$ [deg]	$\cos\phi$	$\sin\phi$	$\cos 2\phi$	$\sin 2\phi$			
160018.183	9168.08	15	0.125	-45	0.707	-0.707	0	-1			
$b_{11i}$	$b_{12i}=b_{21i}$	$b_{22i}$	$b_{13i}=b_{31i}$	$b_{23i}=b_{32i}$	$b_{33i}$	B11	B12	B22	B13	B23	B33
48746.8182	39146.82	48746.82	-37712.53	-37712.53	40646.31	91400.28	73400.284	91400.28	-70711	-70711	76211.83

**Table 18 Plies elastic constants,  $-45^\circ$  plies**

Elastic constants											
$E1_i$ [MPa]	$E2_i$ [MPa]	$n$ [-]	$\delta$ [mm]	$\phi$ [deg]	$\cos\phi$	$\sin\phi$	$\cos 2\phi$	$\sin 2\phi$			
160018.183	9168.08	10	0.125	0	1	0	1	0			
$b_{11i}$	$b_{12i}=b_{21i}$	$b_{22i}$	$b_{13i}=b_{31i}$	$b_{23i}=b_{32i}$	$b_{33i}$	B11	B12	B22	B13	B23	B33
160018.183	3300.51	9168.076	0	0	4800	200022.7	4125.6341	11460.09	0	0	6000

**Table 19 Plies elastic constants,  $0^\circ$  plies**

Laminate elastic constants								
$E_x$ [MPa]	$E_y$ [MPa]	$\mu_{xy}$	$\mu_{yx}$	$G_{xy}$ [MPa]	$\eta_{xy,x}$	$\eta_{xy,y}$	$\eta_{x,xy}$	$\eta_{y,xy}$
53112.95	26951.75	0.777	0.394	31684.7332	0	0	0	0

Membrane stiffness							
B[N/mm]	B11[N/mm]	B12[N/mm]	B22[N/mm]	B13[N/mm]	B23[N/mm]	B33[N/mm]	$\delta\Sigma$ [mm]
8.17E+15	382823.30	150926.20	194260.66	0	0	158423.666	5

**Table 20 Laminate properties**

Laminate loads		
$\sigma_x$ [MPa]	$\sigma_y$ [MPa]	$\tau_{xy}$ [MPa]
-323.38	0	-370.29

**Table 21 Laminate stresses**

Ply effective stress									
a11i	a12i	a13i	a21i	a22i	a23i	a31i	a32i	a33i	$\phi$ [deg]
2.1E-06	1.12E-05	1.58E-05	2.10E-06	1.12E-05	-1.58E-05	0	0	0	45
$\sigma_{_1i}$ [MPa]	$\sigma_{_2i}$ [MPa]	$\tau_{_12i}$ [MPa]							
-1026.66	25.82	52							

**Table 22 Ply stress in local CS 12, 45° plies**

Ply effective stress									
a11i	a12i	a13i	a21i	a22i	a23i	a31i	a32i	a33i	$\phi$ [deg]
2.1E-06	1.12E-05	-1.58E-05	2.10E-06	1.12E-05	1.58E-05	0	0	0	-45
$\sigma_{_1i}$ [MPa]	$\sigma_{_2i}$ [MPa]	$\tau_{_12i}$ [MPa]							
804.85	-42.75	-52							

**Table 23 -45°**

Ply effective stress									
a11i	a12i	a13i	a21i	a22i	a23i	a31i	a32i	a33i	$\phi$ [deg]
1.883E-05	-1.46E-05	0	-1.46E-05	3.71E-05	0	0	0	0	0
$\sigma_{_1i}$ [MPa]	$\sigma_{_2i}$ [MPa]	$\tau_{_12i}$ [MPa]							
-958.65	23.27	-56							

**Table 24 0°**

ply orientation[deg]	RF_1	RF_2	RF_3
0°	1.10	3.14	2.57
45°	2.60	5.22	2.57
-45°	1.17	3.48	2.38

**Table 25 Ply reserve factors**

Resultant laminate stacking sequence is  $[\pm 45_2/(\pm 45/0)_5]_S$ , overall thickness  $t=5\text{mm}$ .

### 3.9 FE stress analysis of the slat

#### 3.9.1 FE model description

Slat FE model was created for stress analysis. Model was created in ANSA software by shell elements. Model geometry was taken from Catia CAD model. MAT8 material model was chosen for the composite ply. Laminate was defined by PCOMP property type. Boundary conditions were defined by SPC1 elements, connections by RBE2 elements. Aerodynamic loads were applied at the cross-sectional centers of pressure as resultant nodal forces. Inertia forces were applied as an acceleration.

Model attachments are defined in 3 points:

- Point 1 - 1,2,3,6 DOF's were taken in local CS system
- Point 2 - 1,2,6 DOF's were taken
- Point 3 - 1,2,6 DOF's were taken

Loads were taken from [6]

### 3.9.2 Slat FE simulation

#### 3.9.2.1 Static stress analysis

Stress analysis of the structure developed in chapter 3.7 was done in NX NASTRAN v.9.0. Solver type – SOL101, Linear Static.

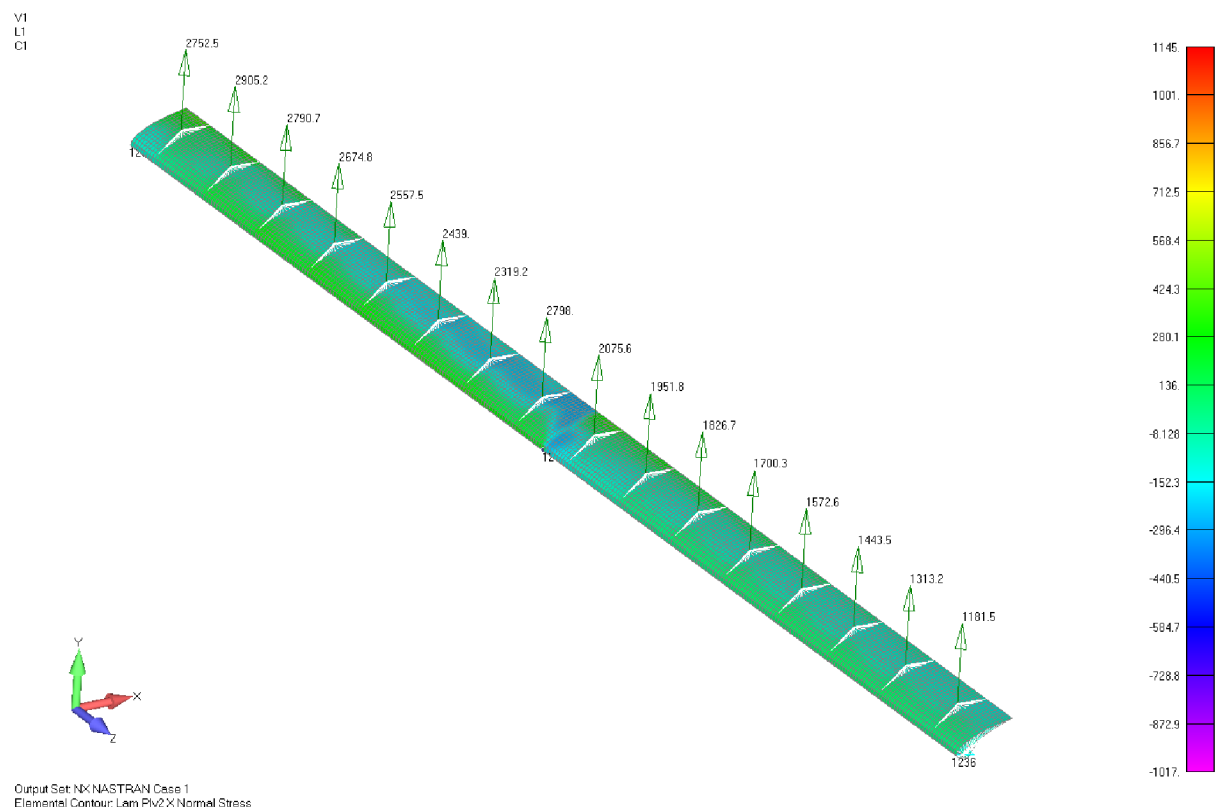
Post processing was done in FEMAP software. The analysis has shown that the skin thickness was overestimated by the analytical calculations and could be reduced. Number of  $\pm 45^\circ$  layers was decreased to 6,  $0^\circ$  layers to 8. Resultant laminate stacking sequence is  $[(\pm 45/0)_3/0]_s$ , overall thickness  $t$  is 2.5mm. The FE analysis of the structure with the new stacking sequence showed that max. stresses occur in ply No. 2. Results are shown in Tables 26, 27 and figures 30 - 32 below.

orientation	$\sigma_t$ [MPa]	$\sigma_c$ [MPa]	$\tau$ [MPa]
L	1145	1017	57.51
T	48.48	27.17	

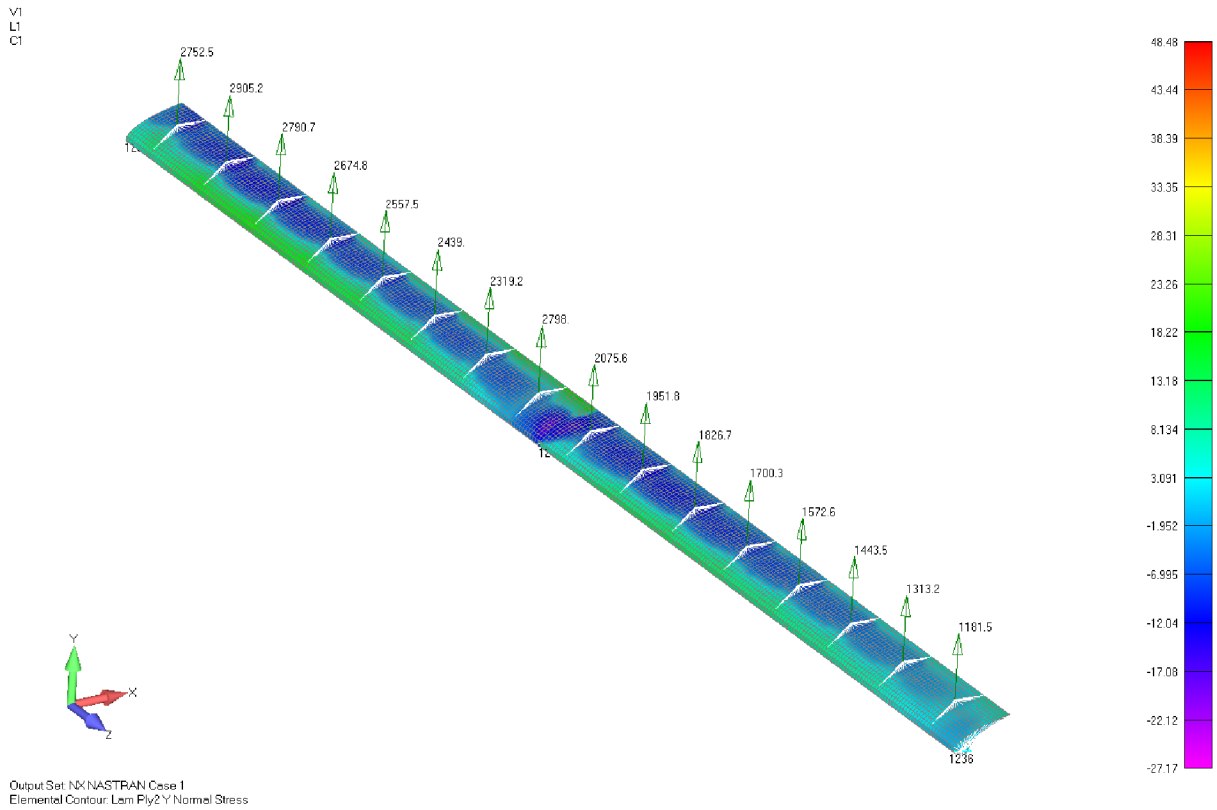
**Table 26 Ply 2 stresses**

orientation	RF_1	RF_2	RF_3
L	1.83	1.11	2.32
T	1.67	8.21	

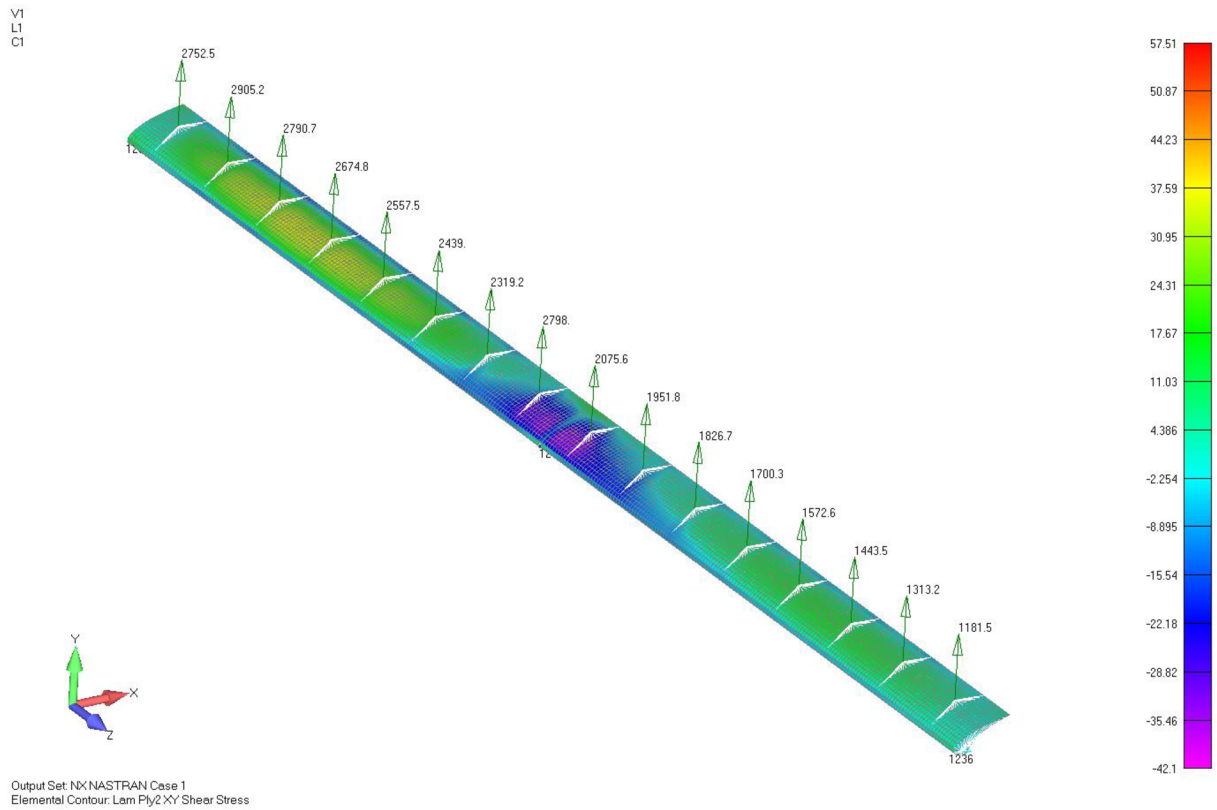
**Table 27 Ply 2 reserve factors**



**Figure 30 Ply 2 stresses, longitudinal**



**Figure 31 Ply 2 stresses, transversal**

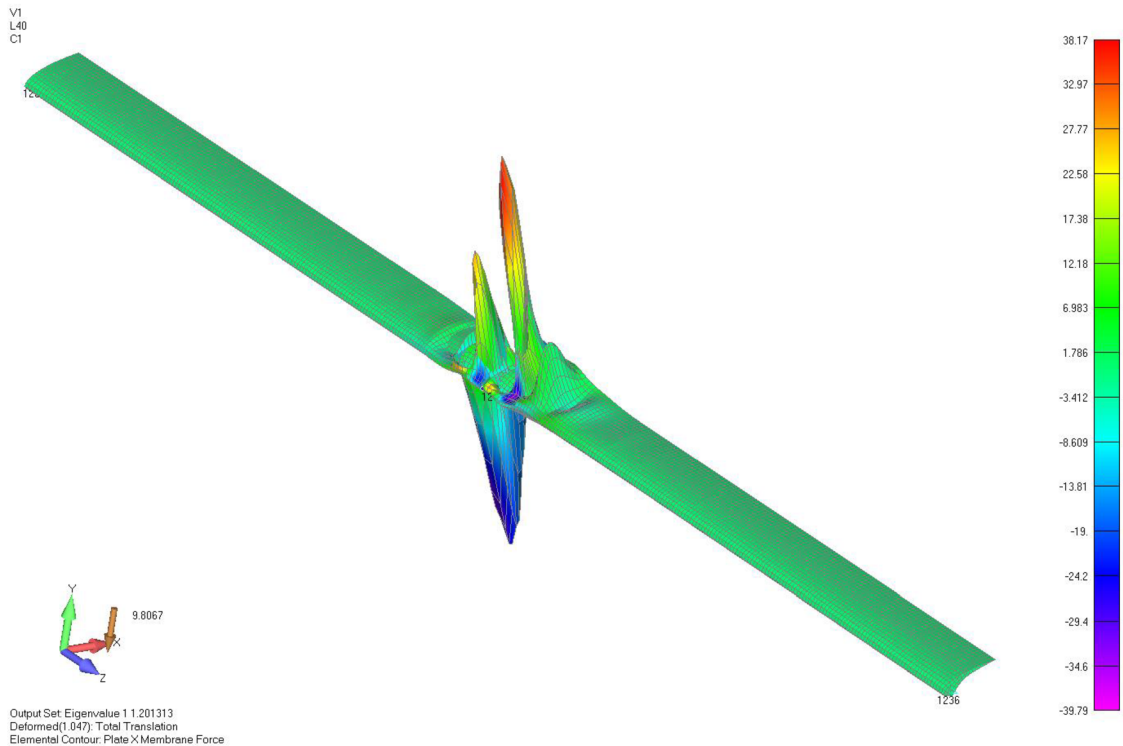


**Figure 32 Ply 2 shear stresses**

### 3.9.2.2 Buckling analysis

Buckling analysis was performed in NX NASTRAN v.9.0 also. Solver type – SOL105, Linear Buckling Analysis.

The analysis has shown that the structure loses stability at 86% of applied loads (Eigen number  $\lambda$  is 0.86). Therefore, two 0° plies have been added to the skin. As a result, minimum Eigen number  $\lambda$  grew up to 1.201 (see Figure 33). That means the structure will be stable at applied loads.



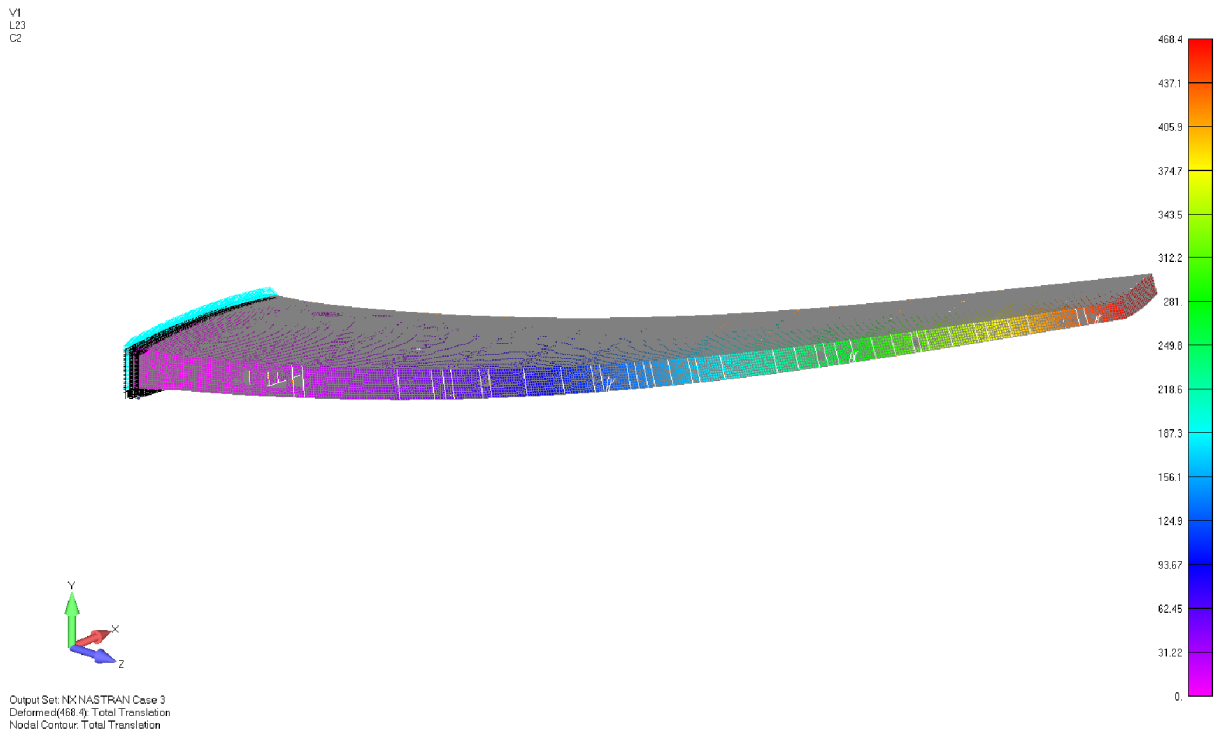
**Figure 33 Slat X Membrane force, Eigenvalue  $\lambda = 1.201$**

### 3.10 Wing deflection influence

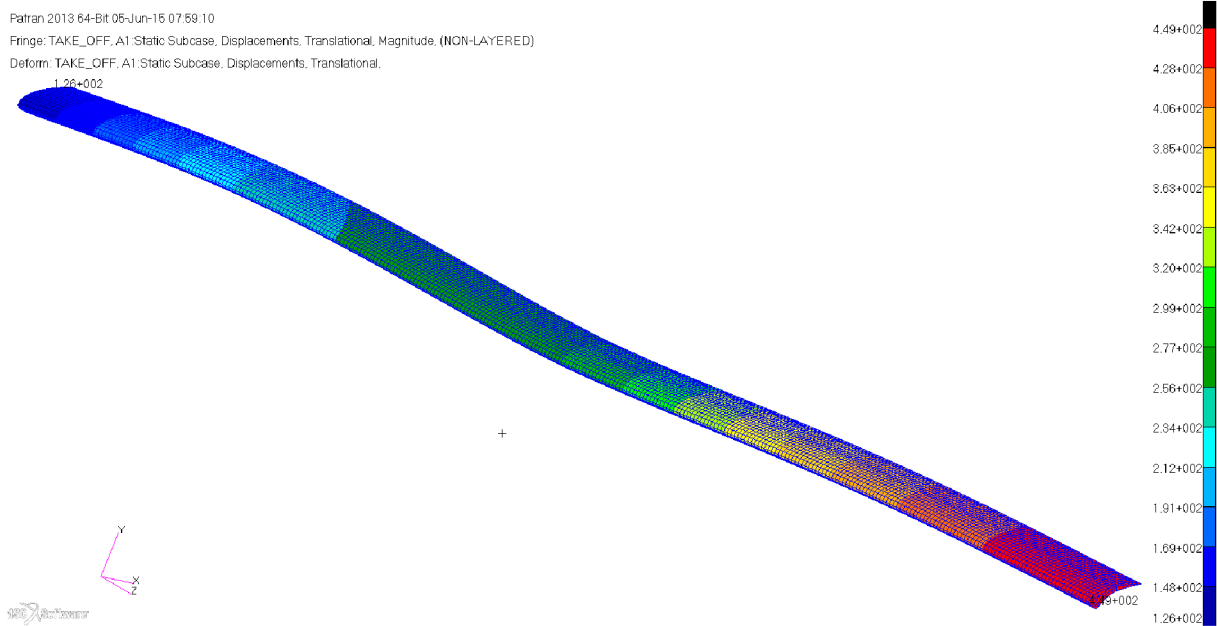
In order to assess slat behavior with respect to a wing deflection during corresponding phase of the flight, slat attachments translations have been measured. It was done by loading of the wing with aerodynamic forces for three load cases (take-off, cruise, landing). Attachments were connected to the wing by MPC2 and CROD elements. Based on these translations the slat was loaded again by aerodynamic forces. Slat deflections were measured according to the wing deflection. Maximum wing deflection and corresponding slat deflections are showed on Figure 34, 35 below.

att. ID[-]	Translation		
	x[mm]	y[mm]	z[mm]
131676	104	437	13
131677	62.6	267	12
131680	28.2	123	10

**Table 28 Attachments translations, take-off load case**



**Figure 34 Wing deflection, take-off load case**



**Figure 35 Slat deflections, take-off load case**



### 3.11 Slat construction recalculation

Simulation made in chapter 3.10 showed that maximum slat deflection (50.9mm) is too high. In order to decrease its value it is needed to increase construction stiffness. As a way to do that increasing number of attachments was chosen. All previous analysis made in chapter 3.9 were done again. Slat frame and rib thickness decreased to 1.5mm.

Maximum stresses occur in Ply 1. Stresses and reserve factors are tabulated below.

orientation	$\sigma_t$ [MPa]	$\sigma_c$ [MPa]	$\tau$ [MPa]
L	1147	1003	51.94
T	62.55	49.38	

**Table 29 Ply 1 stresses**

orientation	RF_1	RF_2	RF_3
L	1.83	1.12	2.6
T	1.29	4.52	

**Table 30 Ply 1 reserve factors**

Maximum slat deflection is 11 mm.

### 3.12 Slat construction summary

After completing analytical calculation and numerical simulation, it was decided:

- Slat frame thickness is 1.5 mm
- Result frame laminate structure is  $[(\pm 45/0)_2]_s$
- Slat ribs thickness is 1.5 mm
- Result rib laminate structure is  $[\pm 45]_{3s}$ .

FEM model has been upgraded according to revised parameters.

## 4 Bird strike simulation

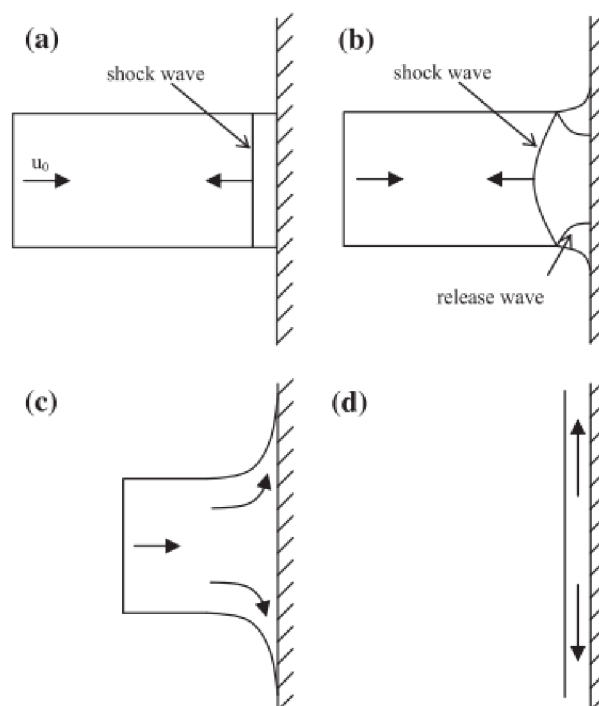
Bird strike incidents cause significant flight safety threats to flying aircraft. Only in the States of America, each year, 36,000 aircraft accidents, produced by bird-strike, are estimated, and since 1988, wildlife strikes have killed more than 194 people and destroyed over 163 aircrafts. In the US, 92% of the strikes occur at below 3000 feet (920m) and 97% of the reported strikes occur during the taking off and landing phase of the aircraft [13]. Population development of large flocking birds has increased dramatically in many parts of the world. Nowadays bird-strike becomes a design requirement. Bird strike analysis involves nonlinear dynamics (material and geometric), contact/coupling, failure mods, large displacements, and other complexities [14]. During bird-strike investigations, it is needed to focus on:

- Residual strength and stiffness of damage structure.
- Aerodynamic loading on damaged structure.
- Ability of an airplane to continue flight and land safely(“Get Home”)

### 4.1 Bird modelling

Bird is non-homogenous, which is the main limitation in order to obtain repeatable results of tests. According to hydrodynamic theory [15], material strength and the response of the target material to the impact pressures can be neglected, the bird will be considered homogenous to simplify the problem. Schematically the bird impact is shown below.

Wilbeck and Rand [13] conducted that a mixture of 85-90 volume percent water and 10-15% of air can accurately model a real bird analytically, with a slightly increased density for water of  $1.06g/cm^3$ . For this purpose, they recommended that a gelatin bird with 15% porosity (to account for the ads in real birds) represent a real bird accurately.



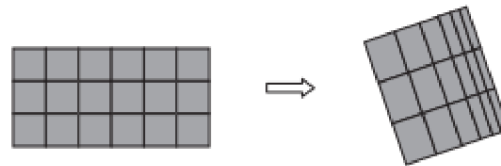
**Figure 36** The phases of bird influence (a) initial impact, (b) shock decay, (c) steady flow, and (d) pressure decay (according to [15])

## 4.2 Bird modelling methods

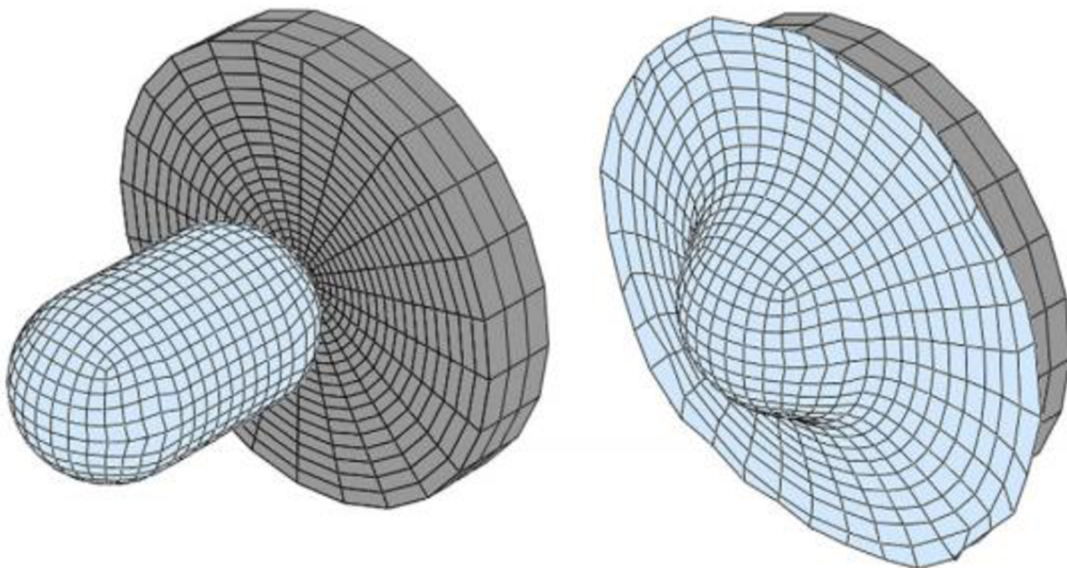
In recent years, explicit FE codes have been used to develop high efficiency bird-proof structures. These codes adopted various finite element approaches to model the impact phenomena: the Lagrangian approach, Eulerian or Arbitrary Lagrangian Eulerian (ALE) approach, and recent solvers based on Smoothed Particle Hydrodynamics (SPH) [16].

### 4.2.1 Lagrangian modelling

The Lagrangian modeling method is the standard approach for most structural finite element analyses. The nodes of the Lagrangian mesh are associated to the material and therefore each node of the mesh follows the material under motion and deformation. This approach is typically used for solid materials. The major problem of Lagrangian bird impactor models are the severe mesh deformations. Large distortions of the elements may lead to inaccurate results, severe hour glassing and even error termination due to negative volume elements. Nowadays this method is considered as an impractical way in bird-strike modelling [16].



**Figure 37 Lagrangian model: nodes are fixed to the material**

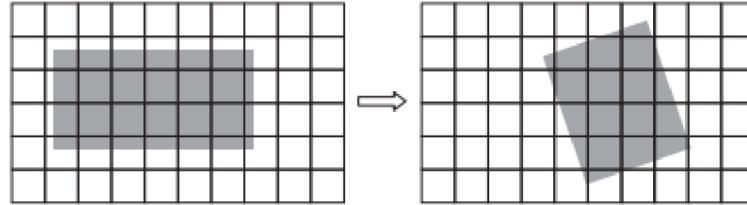


**Figure 38 Bird strike simulation on rigid plate with Lagrangian impactor model**

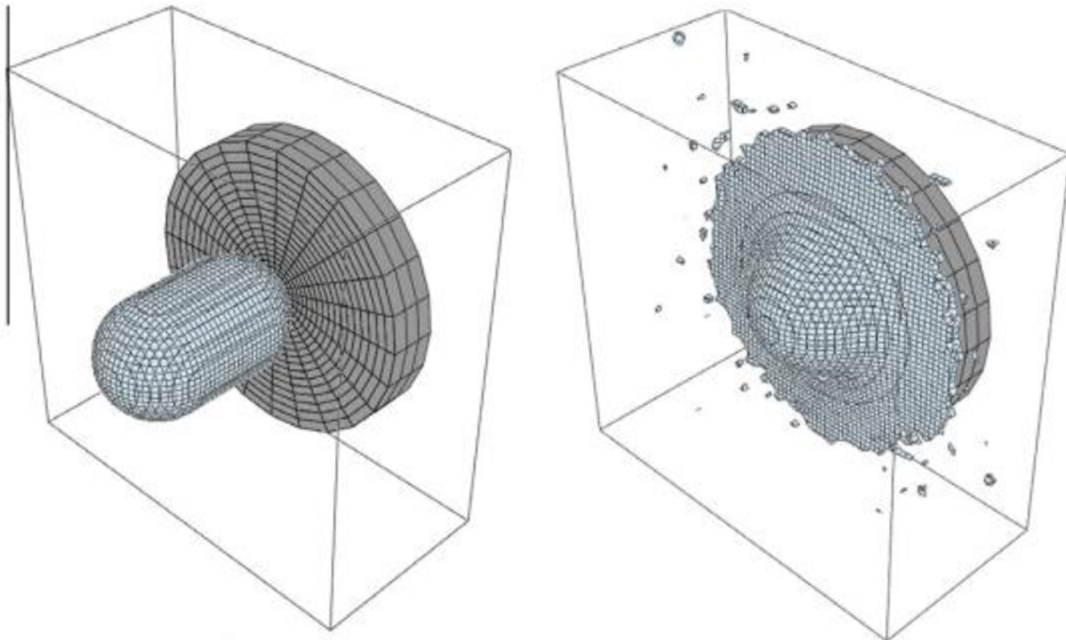
### 4.2.2 Eulerian modelling

In the alternative Eulerian method, the mesh remains fixed in space and the material flows through the mesh. Because the mesh does not move, mesh deformations do not occur and the explicit time step is not influenced. Stability problems due to excessive element deformation do not occur. This approach is typically used for fluid materials and flow processes. Each element has a certain volume fraction of different materials, those can be for example a fluid material and void, or even other materials. This means that each element may be partially filled

with the fluid material. The problem of this solver is numerical leakage, due to dissipation and dispersion problems associated with flux of mass between elements. The computational domain for structural analysis with the Eulerian technique is much larger than with the Lagrangian approach, which leads to high cost of this model, due to the high number of elements and the cost-intensive calculation of element volume fractions [16]. The element size of the Eulerian mesh has to be defined very small in order to achieve accurate results.



**Figure 39 Eulerian model: nodes stay fixed and material flows through the mesh**

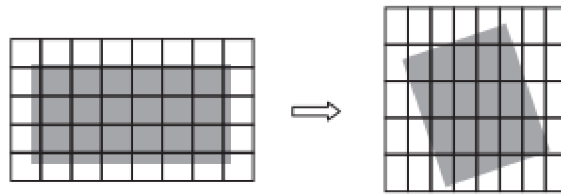


**Figure 40 Bird strike simulation on rigid plate with Eulerian impactor model**

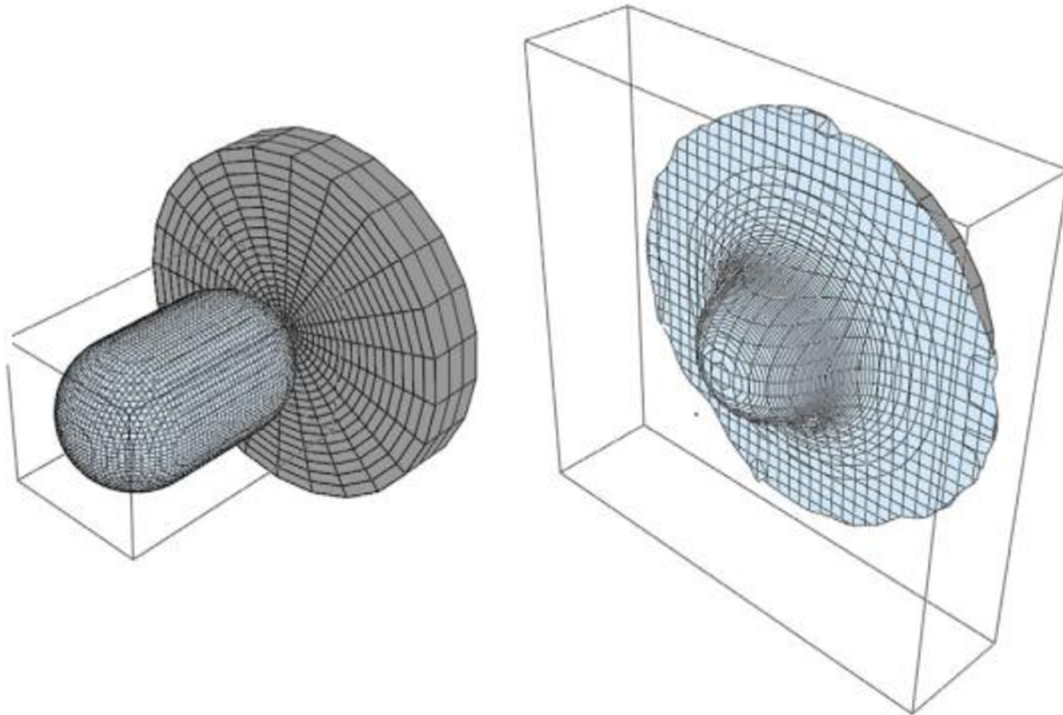
#### **4.2.3 Arbitrary Lagrangian Eulerian (ALE) modelling**

Since in a bird strike simulation typically only the impactor is modeled as a fluid-like body with Eulerian elements and the target as a solid structure with Lagrangian elements, a coupled Eulerian-Lagrangian approach is used for this fluid structure interaction problem. Because the mesh in the classical Eulerian technique is fixed in space, the computational domain should cover not only the region where the material currently exists, but also additional void space to represent the region where material may exist at a later time of interest [16]. In the ALE method the surrounding Eulerian box can move and stretch if needed and is not fixed in space. Results accuracy depends on mesh quality.





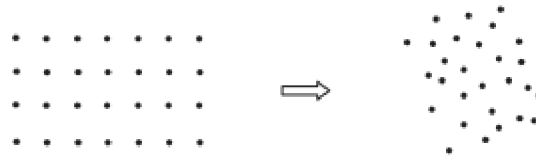
**Figure 41 ALE model: Eulerian mesh moves and deforms with material flowing inside**



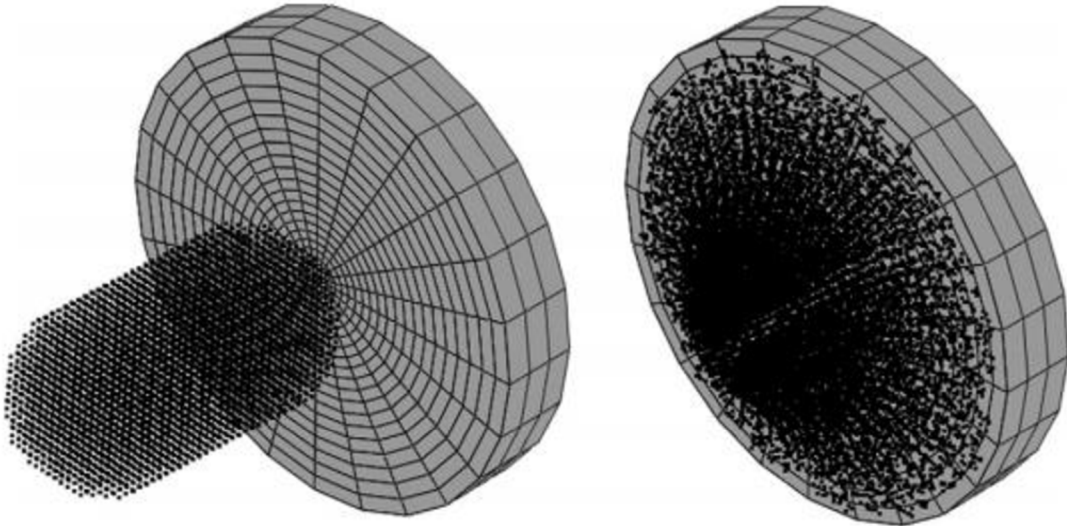
**Figure 42 Bird strike simulation on rigid plate with ALE impactor model**

#### 4.2.4 Smoothed Particle Hydrodynamics (SPH) modelling

The SPH method is a meshless Lagrangian technique, based on interpolation theory and smoothing kernel functions. The fluid is represented as a set of discrete interacting particles (Figure 44), which are independent from each other. Each particle has a mass, velocity and material law assigned to it, which is not localized but smoothed in space by a smoothing kernel function, typically based on a B-spline approximation, defining the range of influence of the particle. Compared to the conventional solid Lagrangian mesh the time step is constant. However, in order to achieve accurate results particle density is required, which needs high memory resources. In comparison to Eulerian modelling the SPH method requires fewer elements, avoids the material interface problems associated with it and normally has a shorter solution time. Disadvantages of the SPH are the lack of sharp boundaries (it is difficult to apply boundary conditions), tension instability (numerical collapse and unphysical clustering of the particles under tension due to negative pressures), undefined impact area (SPH particles do not have a foot print) [16].



**Figure 43 SPH model: fluid is modelled by particles with free motion**

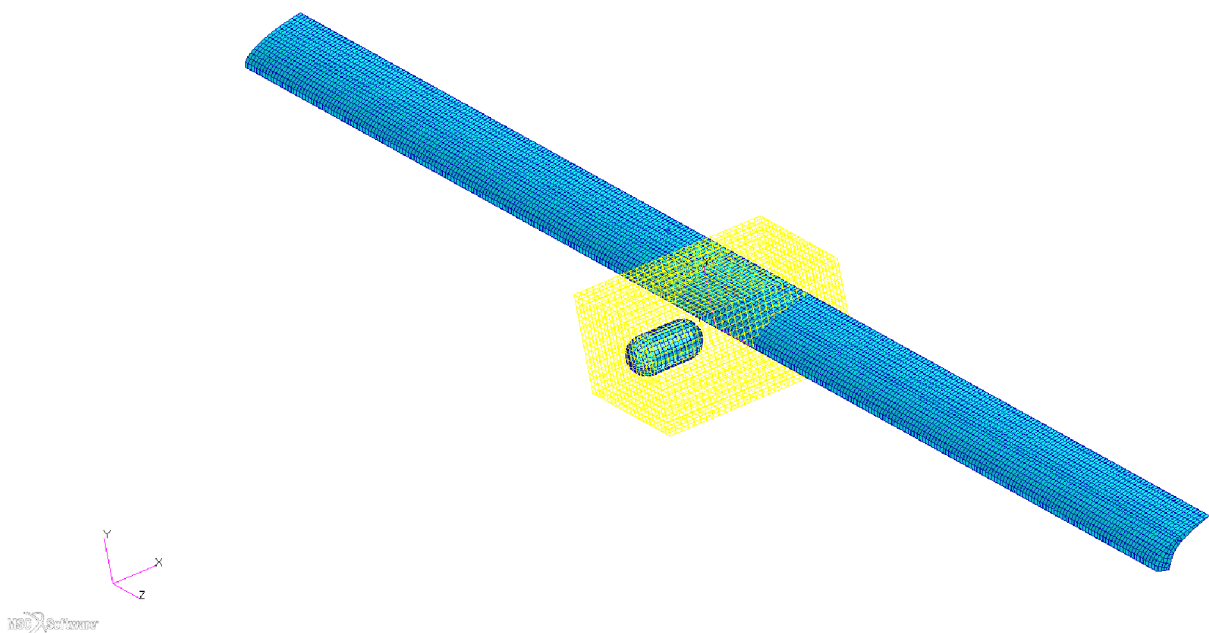


**Figure 44 SPH model**

### **4.3 Bird-strike simulation in MSC.Dytran software**

#### **4.3.1 Simulation conditions**

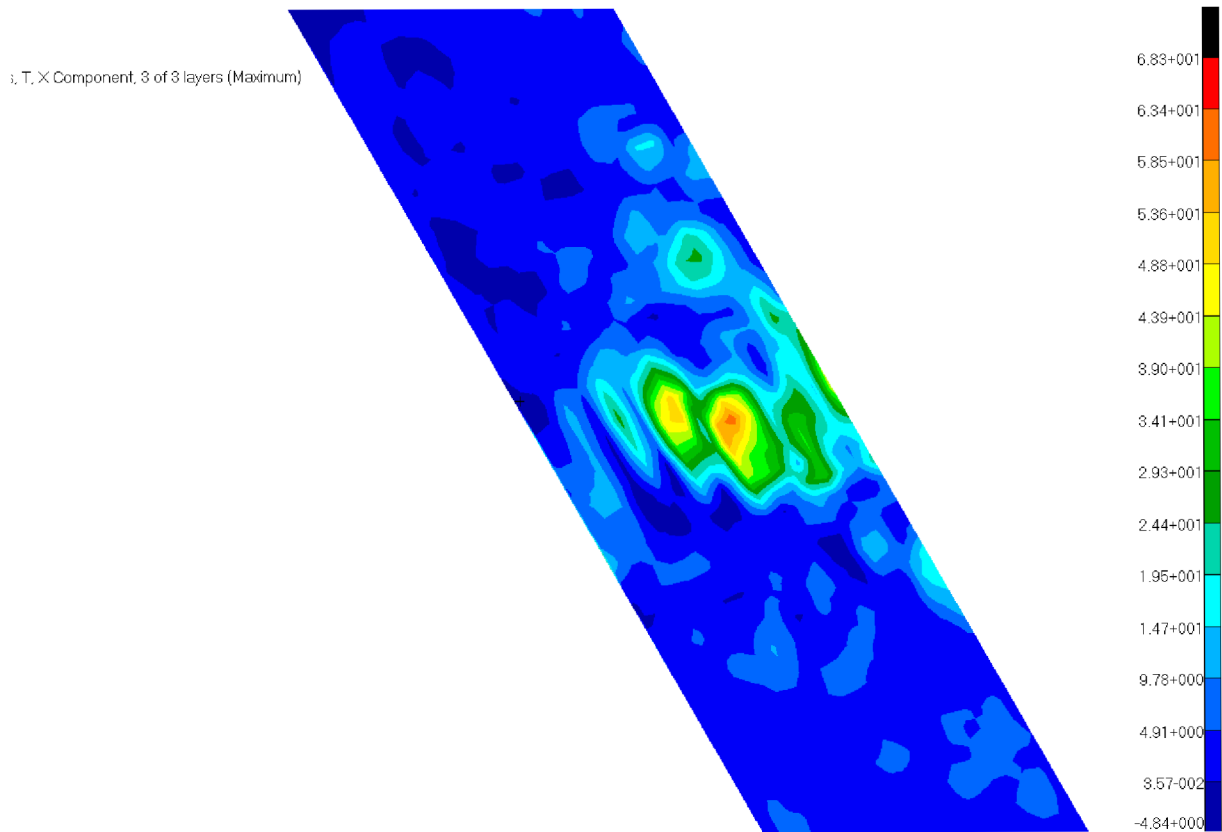
The bird strike simulation was performed using ALE technique. The FE model was created with help of [17], [18]. The slat model was taken from section 3.12.



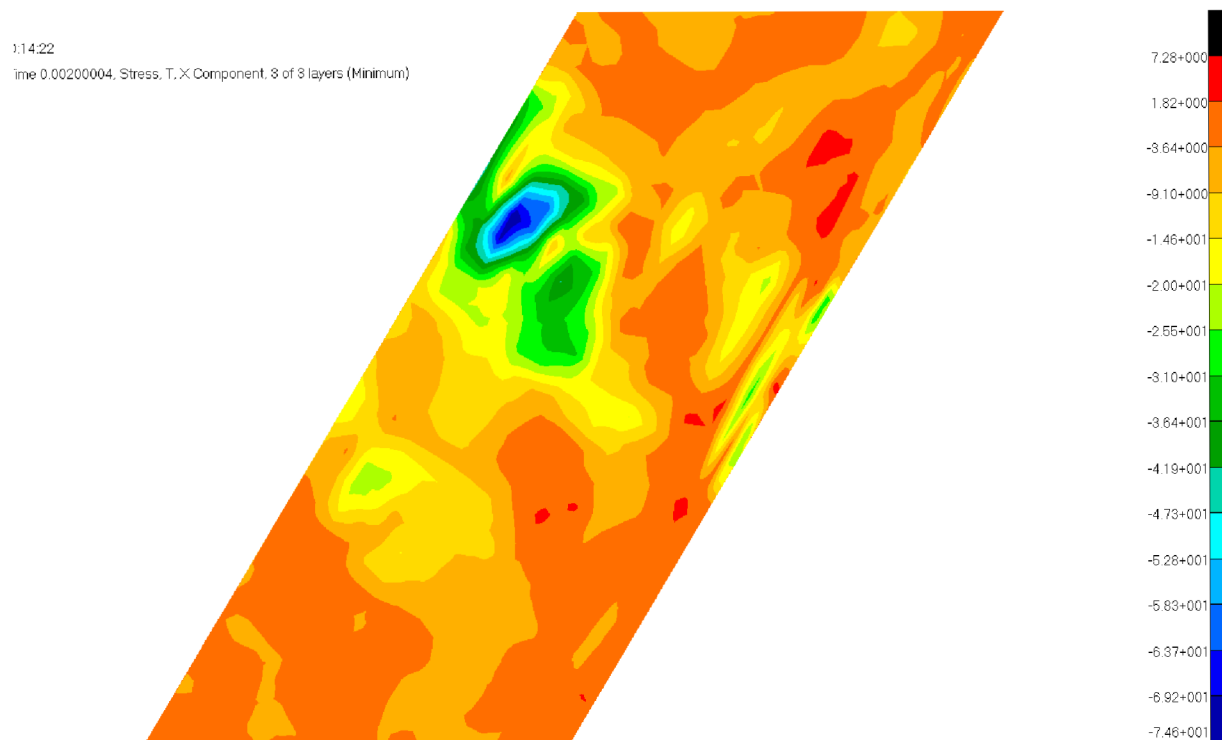
**Figure 45 Bird strike simulation in Dytran**

### 4.3.2 Simulation results

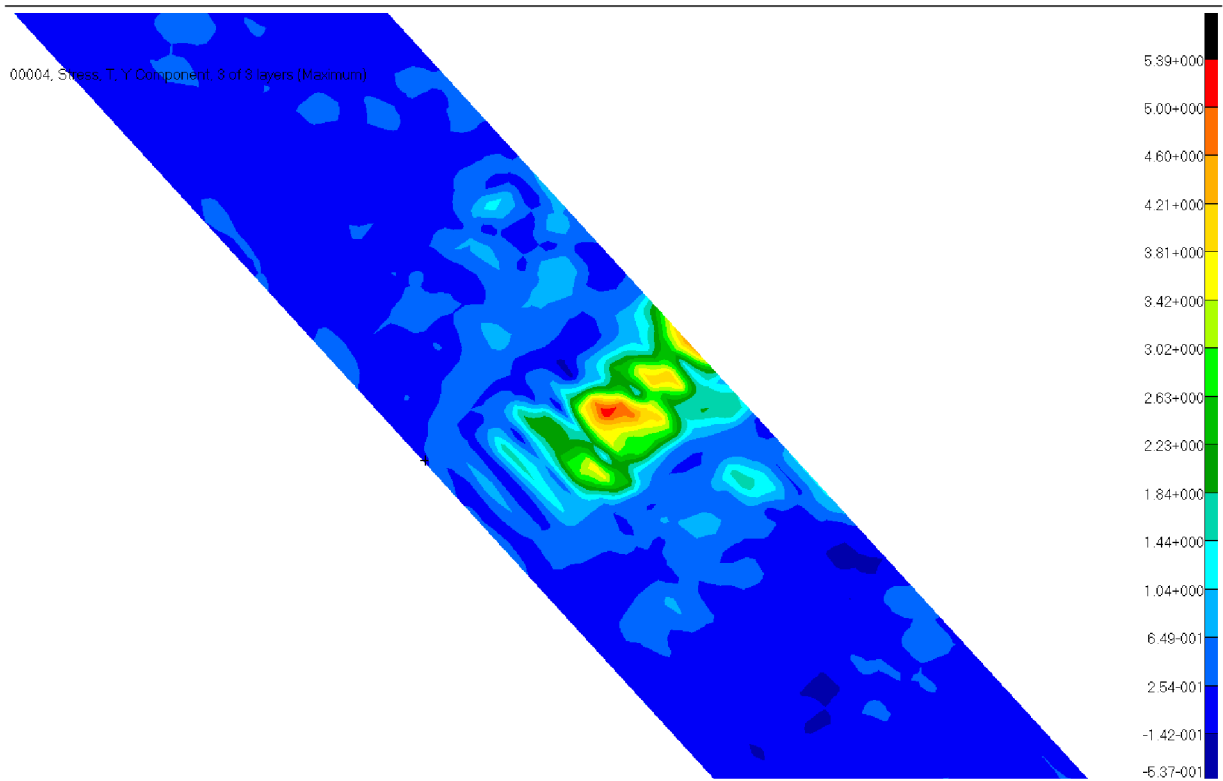
Slat stresses are shown below



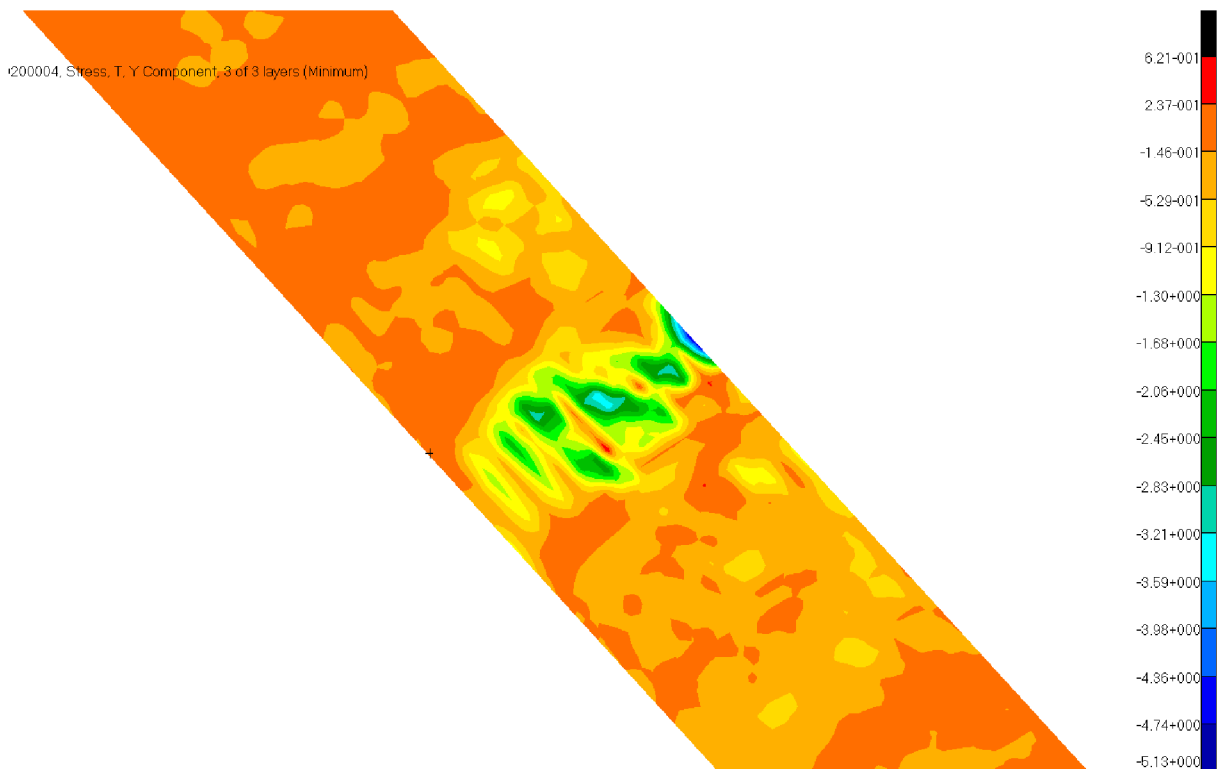
**Figure 46 Slat stresses in tension, longitudinal**



**Figure 47 Slat stresses in compression, longitudinal**

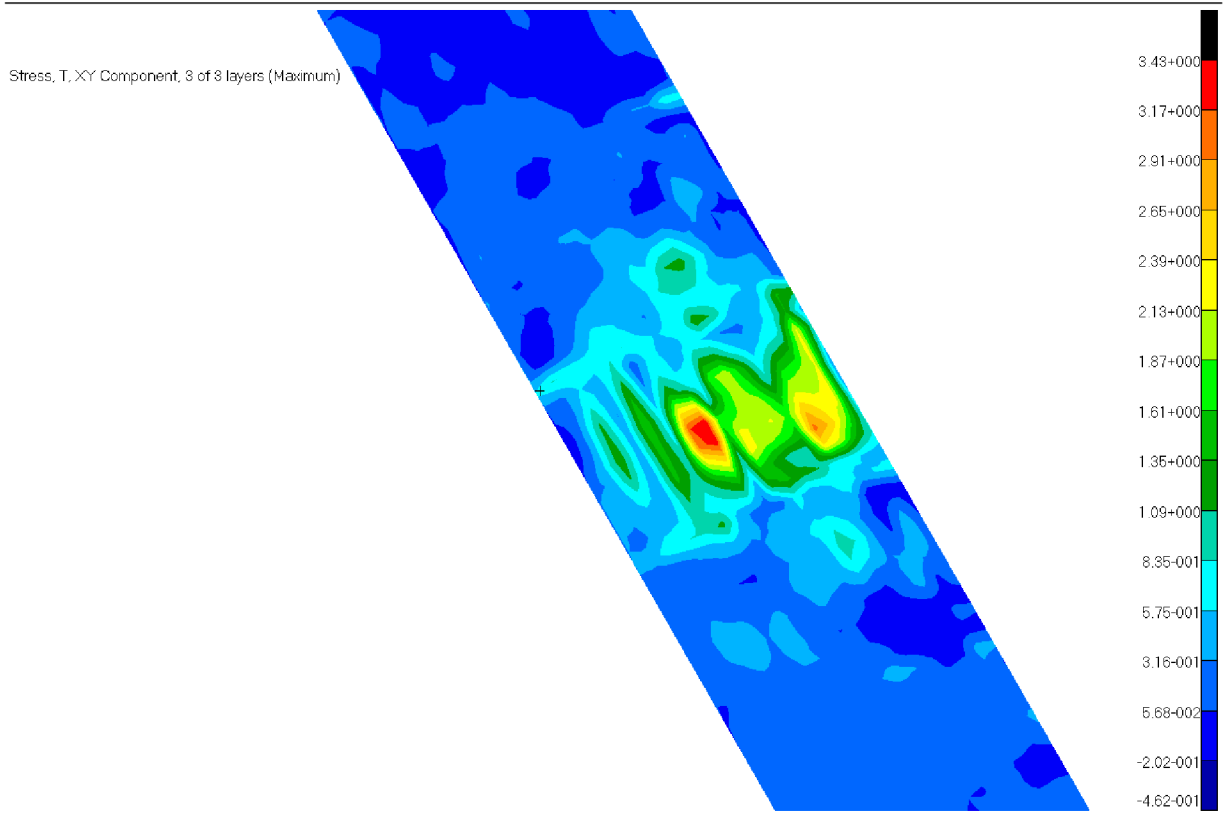


**Figure 48 Slat stresses in tension, transversal**



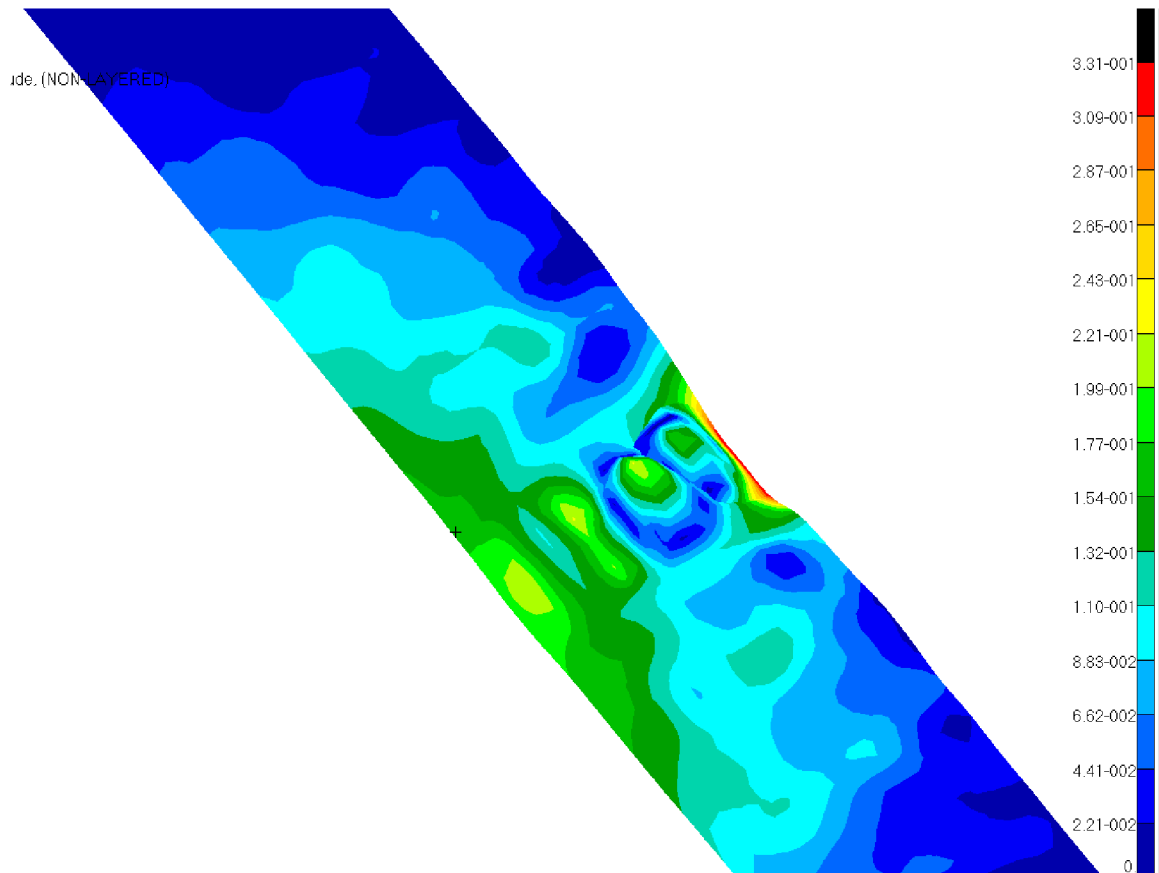
**Figure 49 Slat stresses in compression, transversal**



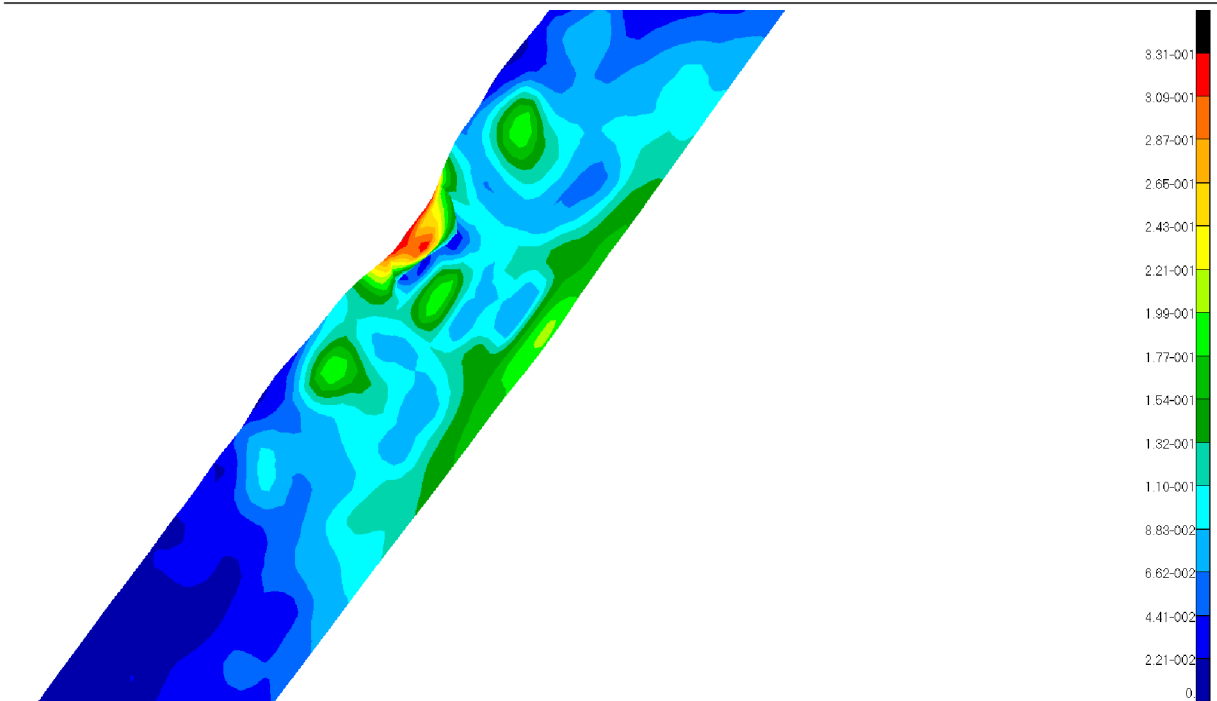


**Figure 50 Slat stresses in shear**

Slat displacements are shown below



**Figure 51 Slat displacements, view from the top**



**Figure 52 Slat displacements, view from the bottom**

#### 4.4 Simulation results assessment

Simulation showed that the bird strike cause relatively small stresses and displacements. Inasmuch the further slat structure modifications are not needed. The resulting slat stresses were calculated as a sum of static and “bird strike” stresses. Note, that the “bird strike” stresses were taken as maximum laminate stresses from inner, outer and middle plies of the skin thickness. This approach is conservative. According to this, the minimum reserve factor is 1.05 that corresponds to fiber compression failure mode. The minimum reserve factor allowed in this thesis is 1.10 but taking into account, that the factor 1.05 was obtained with conservative approach it is considered as allowable.

orientation	$\sigma_t$ [MPa]	$\sigma_c$ [MPa]	$\tau$ [MPa]
L	1215.3	1077.6	55.37
T	31.35	25.14	

**Table 31 Ply 1 result stresses**

orientation	RF_1	RF_2	RF_3
L	1.72	1.05	2.4
T	2.58	8.88	

**Table 32 Ply 1 reserve factors**

## 5 Conclusion

This thesis was focused on design and structural analysis of the composite wing leading edge slat of the Boeing 737-200 aircraft with respect to CS 25 requirements.

Bird strike simulation has shown that the bird strike causes relatively low stresses on the slat and it does not lead to failure of the structure. It is questionable because a real Wilbeck bird impact test made in [7] showed that the forward portion of the leading edge in the contact area was completely destroyed. However, the leading edge analyzed in [7] is rigidly fixed to the torsion box of the wing and cannot move. In turn, the slat designed in this paper is quite elastic and fixed only at five attachment points. Therefore, probably the current slat structure is able to damp the bird strike and not to fail. Animated result fringe proves that the slat structure is bouncing after a collision.

However, in order to make an overall conclusion it is needed to perform extensive amount of simulations, which is beyond the volume of the current work. Moreover, for the final assessment of the slat structure bird strike resistance it is necessary to perform series of full-size real tests. This is also beyond the scope of the current thesis.

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## 9 Appendix

### Analyzed slat model properties

```

Material 1 - 2D ORTHOTROPIC Material
  Type 2D ORTHOTROPIC Color 55 Layer 1 #Prop/Ply 24
    Density 1.45E-9 Damping 0. Ref Temp 0.
    Tsai-Wu 0.
STIFFNESS E1 158830. G12 4800. Nul2 0.36
           E2 9100. G1z 4290.
           G2z 3190.
STRENGTH Tension1 2095.71 Compress1 1126.39 Shear 133.5
          Tension2 81. Compress2 223.2
THERMAL Alpha11 0. K11 0. K12 0.
          Alpha22 0. K22 0. K13 0.
          K33 0. K23 0.
          Spec Heat 0.
OPTICAL Front Off Reverse Off

Property 1 - LAMINATE PLATE Property
  Type LAMINATE PLATE Color 110 Layer 1 Material 0 #Elem 1361
  Laminate Option As Specified
  Failure Theory NONE Bond Shear Allowable 0.
  Ref Temp 0. Damping Coef 0.
  NS Mass/Area 0. Bottom Surf ON 0.75
  Layup 1 -
    Ply 1 Material 1 Thickness 0.125 Angle 45.
    Ply 2 Material 1 Thickness 0.125 Angle 315.
    Ply 3 Material 1 Thickness 0.125 Angle 45.
    Ply 4 Material 1 Thickness 0.125 Angle 315.
    Ply 5 Material 1 Thickness 0.125 Angle 45.
    Ply 6 Material 1 Thickness 0.125 Angle 315.
    Ply 7 Material 1 Thickness 0.125 Angle 315.
    Ply 8 Material 1 Thickness 0.125 Angle 45.
    Ply 9 Material 1 Thickness 0.125 Angle 315.
    Ply 10 Material 1 Thickness 0.125 Angle 45.
    Ply 11 Material 1 Thickness 0.125 Angle 315.
    Ply 12 Material 1 Thickness 0.125 Angle 45.

Property 2 - LAMINATE PLATE Property
  Type LAMINATE PLATE Color 110 Layer 1 Material 0 #Elem 11448
  Laminate Option As Specified
  Failure Theory NONE Bond Shear Allowable 0.
  Ref Temp 0. Damping Coef 0.
  NS Mass/Area 0. Bottom Surf ON 0.75
  Layup 2 -
    Ply 1 Material 1 Thickness 0.125 Angle 45.
    Ply 2 Material 1 Thickness 0.125 Angle 315.
    Ply 3 Material 1 Thickness 0.125 Angle 0.
    Ply 4 Material 1 Thickness 0.125 Angle 45.
    Ply 5 Material 1 Thickness 0.125 Angle 315.
    Ply 6 Material 1 Thickness 0.125 Angle 0.
    Ply 7 Material 1 Thickness 0.125 Angle 0.
    Ply 8 Material 1 Thickness 0.125 Angle 315.
    Ply 9 Material 1 Thickness 0.125 Angle 45.
    Ply 10 Material 1 Thickness 0.125 Angle 0.
    Ply 11 Material 1 Thickness 0.125 Angle 315.
    Ply 12 Material 1 Thickness 0.125 Angle 45.

Load Set 1 - Combined Set
Referenced Sets
  Overall Scale 1.
  Set Scale 1. 10..NASTRAN 10
  Set Scale 1. 40..NASTRAN GRAV 40
Load Set 10 - NASTRAN 10
Load Set 10 - NASTRAN 10
Load Set 10 - NASTRAN 10
  Offset From Absolute Zero 0.
  Stefan-Boltzmann Constant 0.
  Alternate Form Free Convection 0
  Free Convection Exponent 0.
  Alternate Form Forced Convection 0
  Exclude Convective Energy Flow 0
  Fluid Conductivity 0. 0 - None
  Fluid Specific Heat 0. 0 - None
  Fluid Viscosity 0. 0 - None
  Fluid Density 0.
    
```

Loads calculation, stress analysis and bird strike simulation of a composite wing leading edge  
 Institute of Aerospace Engineering, Brno University of Technology

Constant Coefficient 0.  
 Reynolds Number Exponent 0.  
 Prandtl Exponent (into fluid) 0.  
 Prandtl Exponent (out of fluid) 0.

Load Set 10 - NASTRAN 10

Nodal Forces (on Node)

ID	Color	Layer	Def CS	X	Y	Z	Phase
152	10	1	0	0.	1181.536	0.	0.
164	10	1	0	0.	1313.192	0.	0.
176	10	1	0	0.	1443.536	0.	0.
188	10	1	0	0.	1572.566	0.	0.
200	10	1	0	0.	1700.283	0.	0.
212	10	1	0	0.	1826.688	0.	0.
224	10	1	0	0.	1951.779	0.	0.
236	10	1	0	0.	2075.557	0.	0.
248	10	1	0	0.	2798.022	0.	0.
260	10	1	0	0.	2319.174	0.	0.
272	10	1	0	0.	2439.013	0.	0.
284	10	1	0	0.	2557.539	0.	0.
296	10	1	0	0.	2674.753	0.	0.
308	10	1	0	0.	2790.653	0.	0.
320	10	1	0	0.	2905.24	0.	0.
332	10	1	0	0.	2752.459	0.	0.

Load Set 20 - NASTRAN 20

Load Set 20 - NASTRAN 20

Load Set 20 - NASTRAN 20

Offset From Absolute Zero 0.  
 Stefan-Boltzmann Constant 0.  
 Alternate Form Free Convection 0  
 Free Convection Exponent 0.  
 Alternate Form Forced Convection 0  
 Exclude Convective Energy Flow 0  
 Fluid Conductivity 0.  
 Fluid Specific Heat 0.  
 Fluid Viscosity 0.  
 Fluid Density 0.  
 Constant Coefficient 0.  
 Reynolds Number Exponent 0.  
 Prandtl Exponent (into fluid) 0.  
 Prandtl Exponent (out of fluid) 0.

0 - None  
 0 - None  
 0 - None

Load Set 20 - NASTRAN 20

Nodal Forces (on Node)

ID	Color	Layer	Def CS	X	Y	Z	Phase
1764	10	1	0	0.	702.	0.	0.
1781	10	1	0	0.	746.	0.	0.
1798	10	1	0	0.	790.	0.	0.
1836	10	1	0	0.	833.	0.	0.
1853	10	1	0	0.	877.	0.	0.
1870	10	1	0	0.	920.	0.	0.
1925	10	1	0	0.	962.	0.	0.
1989	10	1	0	0.	1004.	0.	0.
2022	10	1	0	0.	1046.	0.	0.
2069	10	1	0	0.	1088.	0.	0.
2150	10	1	0	0.	1130.	0.	0.
2223	10	1	0	0.	1171.	0.	0.
2596	10	1	0	0.	1212.	0.	0.
3062	10	1	0	0.	1252.	0.	0.
3302	10	1	0	0.	1293.	0.	0.
3472	10	1	0	0.	1216.	0.	0.

Load Set 30 - NASTRAN 30

Load Set 30 - NASTRAN 30

Load Set 30 - NASTRAN 30

Offset From Absolute Zero 0.  
 Stefan-Boltzmann Constant 0.  
 Alternate Form Free Convection 0  
 Free Convection Exponent 0.  
 Alternate Form Forced Convection 0  
 Exclude Convective Energy Flow 0  
 Fluid Conductivity 0.  
 Fluid Specific Heat 0.  
 Fluid Viscosity 0.  
 Fluid Density 0.  
 Constant Coefficient 0.  
 Reynolds Number Exponent 0.  
 Prandtl Exponent (into fluid) 0.  
 Prandtl Exponent (out of fluid) 0.

0 - None  
 0 - None  
 0 - None

Load Set 30 - NASTRAN 30

Nodal Forces (on Node)



